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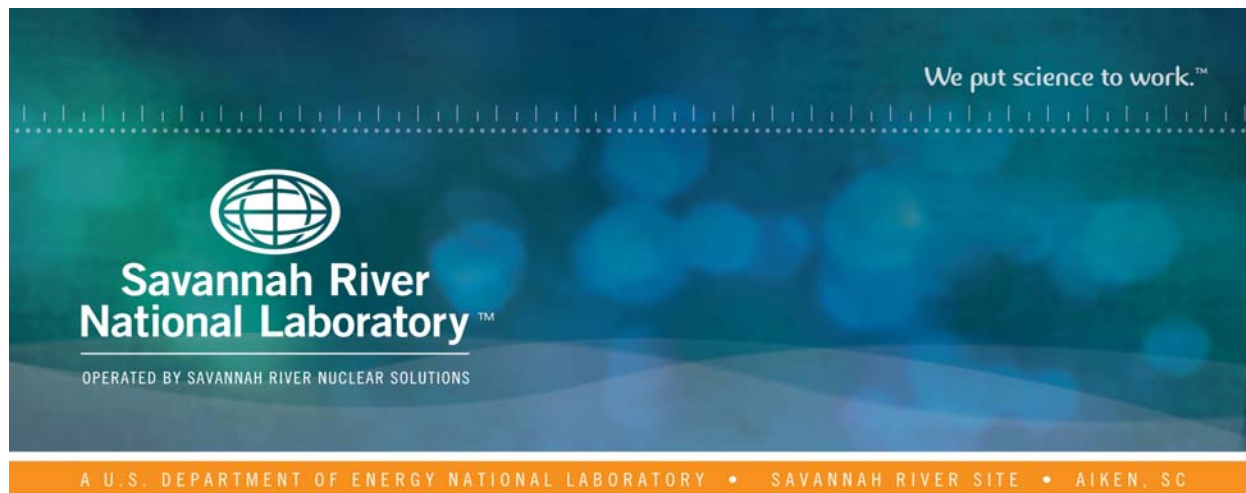
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# Submersible Blend Pump Mixing Evaluation

M. R. Poirier

April 2019

SRNL-STI-2019-00176, Revision 0



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## EXECUTIVE SUMMARY

Savannah River Remediation (SRR) requested Savannah River National Laboratory (SRNL) to perform an engineering analysis to investigate the mixing behavior in feed blend tanks for the Salt Waste Processing Facility (SWPF). The baseline for mixing the feed blend tanks is a  $U_0D$  of 5.1 ft<sup>2</sup>/s, a pump nozzle elevation of 165.86 inches above the sludge layer and 174 inches above the tank bottom, a liquid level 174.28 inches above the pump nozzle, and a blend time of 10.63 hours. The current Salt Batch Plan identifies Tanks 21, 26, 41, and 42 as the designated blend tanks. This supplemental study only addresses Type IIIA tanks (i.e., Tanks 26, 41, and 42). Using the Tank 41 current configuration as the baseline, the analysis considers the impact on blending if:

- the distance between the pump discharge nozzle and the liquid surface increases above the baseline,
- the distance between the pump discharge nozzle and the liquid surface decreases below the baseline,
- the distance between the pump discharge nozzle and the sludge layer at the bottom of the tank increases above the baseline, or
- the distance between the pump discharge nozzle and the sludge layer at the bottom of the tank decreases below the baseline, each while maintaining the minimum  $U_0D$  of 5.1 ft<sup>2</sup>/s.
  - Per subsequent direction from SRR, the minimum distance between the pump discharge nozzle and the sludge will be 165.86 inches. Hence, distances less than 165.86 inches between the pump discharge nozzle and the sludge were not included in the analysis.
- In addition, SRR requested SRNL to evaluate the impact of the abandoned in place, failed slurry pumps in Tank 42 on liquid blending.

Two approaches were employed to assess the impact of changing the distance between the pump discharge nozzle and the liquid surface and the distance between the pump discharge nozzle and the sludge surface in Tank 41 in preparing salt solution for the SWPF. The approaches are matching the Froude number and examining existing jet blending correlations. The impact of an abandoned in place, failed slurry pump on liquid blending in Tank 42 was evaluated by performing a momentum balance on the flow in the tank to estimate the change in fluid velocity caused by the drag from the abandoned slurry pumps, and its impact on liquid blending.

The conclusions from this study follow.

- Increasing the liquid height in the blend tanks relative to the pump discharge nozzles could increase the blend time. The maximum increase is calculated to be 4 - 9% above the baseline.
- Increasing the pump discharge nozzle elevation relative to the sludge level (or tank bottom) in the blend tanks could increase the blend time. The maximum increase is calculated to be 15 - 33% above the baseline.
- Decreasing the distance between the pump discharge nozzle and the liquid surface to as little as 100 inches is unlikely to adversely impact the liquid blend time, but it could reduce it.
- The abandoned pumps in Tank 42 occupy a very small fraction of the tank, much of the fluid motion in the tank will not be impacted by these abandoned pumps, and the increase in blend time is predicted to be 2.1% above the baseline.

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## LIST OF ABBREVIATIONS

$C_i$	Constants
$C_D$	Drag coefficient
$D$	Pump or jet nozzle diameter
$D_p$	Pump assembly diameter
$F_D$	Drag force
$Fr$	Froude number
$g$	Gravitational acceleration
$H$	Height of tank or fluid
$L$	Length travelled by jet from pump center to tank wall
$L_p$	Length of pump assembly
$\hat{m}$	Jet momentum
$Re$	Reynolds number
SRNL	Savannah River National Laboratory
SRR	Savannah River Remediation
SWPF	Salt Waste Processing Facility
$T$	Tank diameter
TTR	Task Technical Request
$U_i$	Nozzle discharge velocities
$U$	Velocity
$y$	Horizontal distance from pump center to tank wall
$\rho$	Liquid density
$\theta$	Blend Time
$\nu$	Kinematic viscosity



## 1.0 Introduction

Savannah River National Laboratory (SRNL) personnel performed testing utilizing a 1/10.85 pilot-scale model of Tank 50 to determine the specifications for a mixing pump to blend miscible liquids to prepare feed for the Salt Waste Processing Facility (SWPF).<sup>1</sup> One design parameter for opposing, dual nozzle, blending pumps is  $U_0D$ , where  $U_0$  is the discharge velocity for each pump nozzle, and  $D$  is the nozzle diameter. With a liquid waste volume of 1,225,000 gallons (~349 inches of slurry) and a pump elevation of 174 inches (distance between tank bottom and pump discharge nozzle), the prior testing and analysis specified a minimum  $U_0D$  of 5.1 ft<sup>2</sup>/s to ensure the liquids were adequately blended and a maximum  $U_0D$  of 6.1 ft<sup>2</sup>/s to prevent the jets from disturbing and suspending the solid particles settled at the bottom of the Type IIIA Waste Tank.

The pilot-scale testing was performed at a liquid level equivalent to 348.28 inches in a full-scale waste tank. The pump discharge nozzles were the equivalent of 174 inches above the tank bottom in a full-scale waste tank, and the testing contained the equivalent of 8.14 inches of sludge solids on the tank bottom. The equivalent distance between the pump discharge nozzle and the sludge layer is 165.86 inches. The calculated blend time for the feed blend tanks is 10.63 hours with a  $U_0D$  of 5.1 ft<sup>2</sup>/s. The actual installation of the Blend Pump in Tank 41 may result in different liquid levels and discharge nozzle elevations than that modeled.

Savannah River Remediation (SRR) requested SRNL to perform supplemental engineering analysis to investigate the impact on mixing of the blend feed tanks for SWPF. Using the Tank 41 current configuration as the baseline, the analysis considers the impact on blending if:

- the distance between the pump discharge nozzle and the liquid surface increases above the baseline,
- the distance between the pump discharge nozzle and the liquid surface decreases below the baseline,
- the distance between the pump discharge nozzle and the sludge layer at the bottom of the tank increases above the baseline,
- or the distance between the pump discharge nozzle and the sludge layer at the bottom of the tank decreases below the baseline, each while maintaining the minimum  $U_0D$  of 5.1 ft<sup>2</sup>/s (see Attachment 1).<sup>2</sup>

Per direction from SRR, the minimum distance between the pump discharge nozzle and the sludge will be 165.86 inches. Hence, distances less than 165.86 inches between the pump discharge nozzle and the sludge will not be included in the analysis.

The current Salt Batch Plan identifies Tanks 21, 26, 41, and 42 as the designated blend tanks.<sup>3</sup> Tanks 26, 41, and 42 are Type IIIA Tanks with cooling coils. While the details of the tanks, such as blend pump location, riser locations, and cooling coil area may differ slightly, the differences should not impact the required  $U_0D$  for blending these tanks. In addition, SRR asked SRNL to evaluate the impact of abandoned in place, failed slurry pumps in Tank 42 on liquid blending (see Attachment 2). The focus of this analysis is on the Type IIIA tanks (Tanks 26, 41, and 42).

## 2.0 Approach

### 2.1 Changing Distance between Pump Nozzle and Liquid Surface or Sludge Surface

Two approaches were employed to assess the impact of changing the distance between the pump nozzle and the liquid surface or the sludge surface in the blend tanks (Tanks 26, 41, and 42) for SWPF. The approaches are matching the Froude number and examining existing jet mixing correlations.

### 2.1.1 Equal Froude Number

In analyzing and modeling the effects of changes in distance between the pump discharge nozzle and the liquid surface on mixing processes, a Froude number is often used.<sup>4</sup> The Froude number is the ratio of inertial forces to gravitational forces, and is described by equation [1]

$$Fr = \frac{U^2}{g \cdot H} \quad [1]$$

where  $U$  is the fluid velocity,  $g$  is the acceleration due to gravity, and  $H$  is the fluid height above the discharge nozzle.

To maintain equal liquid surface motion for two different levels, the Froude number is assumed to be constant. For constant Froude number and gravitational acceleration, the relationship between fluid height and nozzle discharge velocity is described by equation [2].

$$U_1^2 \cdot H_2 = U_2^2 \cdot H_1 \quad [2]$$

In equation [2],  $U_1$  and  $H_1$  refer to the nozzle discharge velocity and liquid height for the baseline conditions, and  $U_2$  and  $H_2$  refer to the nozzle discharge velocity and liquid height for the modified conditions. Given that mixer pump discharge nozzle diameter is constant, multiplying equation [2] by  $D^2$  and solving for  $U_2 D$  yields equation [3].

$$U_2 D = U_1 D \sqrt{\frac{H_2}{H_1}} \quad [3]$$

The baseline condition is  $U_1 D$  of 5.1 ft<sup>2</sup>/s, liquid level of 348.28 inches, and pump elevation of 174 inches; hence  $H_1$  is 174.28. If the distance between pump discharge nozzle and the liquid surface is increased to 190 inches, the  $U_2 D$  required for equal surface motion is calculated in equation [4].

$$U_2 D = 5.1 \frac{\text{ft}^2}{\text{s}} \sqrt{\frac{190 \text{ in}}{174.28 \text{ in}}} = 5.3 \frac{\text{ft}^2}{\text{s}} \quad [4]$$

Equation [4] shows that an ~4% increase in  $U_0 D$  would be required to have equivalent mixing at the liquid surface. If the  $U_0 D$  is maintained at 5.1 ft<sup>2</sup>/s, the blend time will increase, slightly. Equation [5] is a general expression for blend time in a jet mixed tank

$$\theta = C_1 \frac{f(T, H)}{U_0 D} \quad [5]$$

where  $C$  is a constant and  $T$  is the tank diameter.<sup>5</sup> Since blend time is inversely proportional to  $U_0 D$  and  $U_0 D$  is not being increased, increasing the height will increase the blend time approximately 4%.

### 2.1.2 Miscible Liquid Blend Time Correlations

Another approach to assess the impact of changing liquid height and pump elevation on liquid blending is to look at the impact of liquid height in blend time correlations. The correlations were developed for open tanks with no cooling coils. However, the testing performed by SRNL showed a similar influence of  $U_0 D$  on blend time with and without coils, with a different constant to account for the cooling coils.<sup>1</sup> One correlation that includes the impact of liquid height on blend time is described by equation [6]<sup>6</sup>

$$\theta = C_2 \frac{T^{1.5} H^{0.5}}{U_0 D} \quad [6]$$

where  $\theta$  is blend time. Equation [6] shows that the blend time increases with the square root of the change in tank liquid elevation. Since the mixer pump is located in the middle region of the tank rather than the bottom of the tank, the height used in equation [6] is the distance between the pump nozzle and the liquid surface or the difference between the pump nozzle and the sludge layer on the bottom of the tank.

Increasing the distance between the pump discharge nozzle and the liquid surface from 174.28 inches to 190 inches, is a 9% increase. Since the blend time is proportional to the square root of the height (equation [6]), this increase would lead to an ~4% increase in blend time.

Increasing the distance between the pump discharge nozzle and the sludge layer from 165.86 inches to 220 inches, is a 33% increase. Since the blend time is proportional to the square root of the height, this increase would lead to an ~15% increase in blend time.

Fox and Gex (equation [7]) and Lane and Rice (equation [8]) developed correlations that show the same dependence of mixing time on height, but a different influence of  $U_0 D$  as compared to equation [6].<sup>7</sup>

$$\theta = C_3 \frac{TH^{0.5}}{(U_0 D)^{2/3}} \quad [7]$$

$$\theta = C_4 \frac{TH^{0.5}}{(U_0 D)^{2/3}} \quad [8]$$

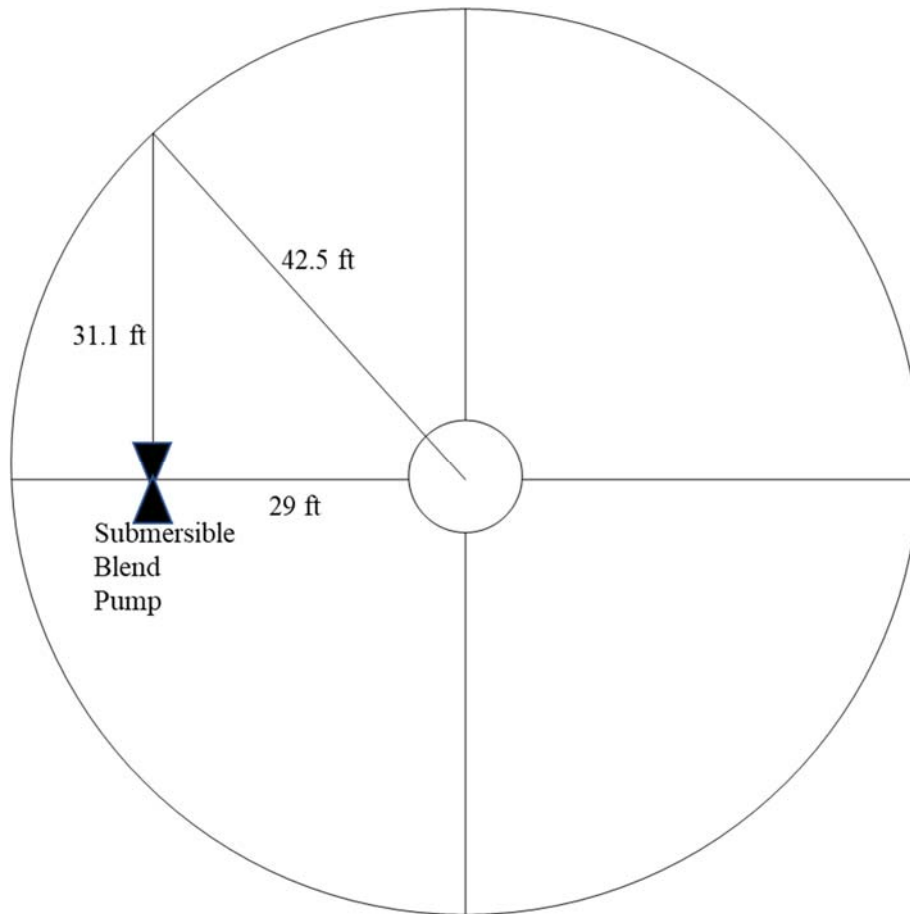
Using equations [7] and [8], increasing the distance between the pump discharge nozzle and the liquid surface from 174.28 to 190 inches would increase the blend time by ~4%, and increasing the distance between the pump discharge nozzle and the sludge surface from 165.86 inches to 220 inches, would increase the blend time by 15%, consistent with the results from equation [6], given  $U_0 D$  does not change.

Grenville and Tilton developed a correlation to predict the blend time in a jet mixed tank. Their correlation is described by equation [9].<sup>8</sup>

$$\theta = C_5 \frac{TH}{(U_0 D) L} \quad [9]$$

In equation [9],  $L$  is the distance travelled by the jet prior to hitting the wall of the tank. The submersible blend pump discharge nozzle is located 29 feet from the center of the tank having a radius of 42.5 feet.<sup>9,10</sup> The horizontal distance between the blend pump center and the tank wall ( $y$ ) is described by equation [10] and Figure 1.

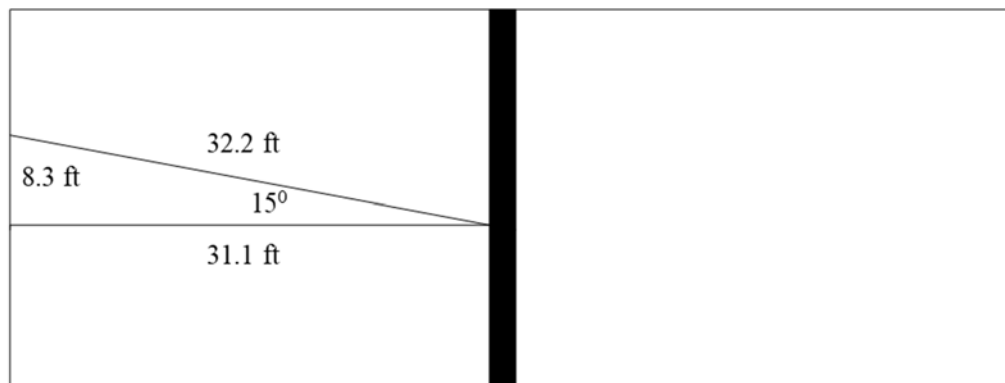
$$29^2 + y^2 = 42.5^2 \quad [10]$$



**Figure 1. Layout of Submersible Blend Pump in Type IIIA Waste Tank**

Solving equation [10] for  $y$  gives a distance of 31.1 feet. Because the pump discharge nozzles are angled  $15^\circ$  above the horizontal plane, the distance ( $L$ ) travelled by the jet prior to hitting the wall of the tank is longer than 31.1 feet and is described by equation [11] and Figure 2.

$$\frac{31.1 \text{ ft}}{L} = \cos 15^\circ \quad [11]$$



**Figure 2. Side View of Pump Discharge Jet**

Solving equation [11] for L, gives a value of 32.2 feet. The height above the nozzle at which the pump discharge jet impacts the wall is 8.3 feet, or 99.6 inches. The value of L will not change, but if this distance is greater than the distance between the pump discharge nozzle and the liquid surface, it would shorten the path length of the jet produced by the pump and increase the blend time. Since the minimum distance between the pump discharge nozzle and the liquid surface is 100 inches, the only impact of decreasing this distance from 165.86 inches to 100 inches would be to decrease the blend time. Because the discharge of the jet is not at the center of the pump assembly, the distance between the discharge at the jet nozzle to the tank wall will be slightly less than 32.2 feet, and the height above the nozzle at which the pump discharge jet impacts the wall will be slightly less than 99.6 inches.

For mixing at the liquid surface, increasing the height from 174.28 inches to 190 inches, increases the blend time by 9% compared with the baseline, according to equation [9].

For mixing at the sludge surface, increasing the distance between the pump discharge nozzle and the sludge surface from 165.86 inches to 220 inches, increases the blend time by 33% compared with the baseline, according to equation [9].

Applying the blend time correlations to assess the impact of increasing the distance between the pump discharge nozzle and the liquid surface shows the increase in blend time will range from 4% to 9%, compared with the baseline. Applying the blend time correlations to assess the impact of increasing the distance between the pump discharge nozzle and the sludge layer shows the increase in blend time will range from 15% to 33%, compared with the baseline.

## 2.2 Abandoned in Place Failed Slurry Pumps

Per the Task Technical Request (TTR), the submersible blend pump will be installed in riser B4 (see Attachment 2). The submersible blend pump has two discharge nozzles, with a nozzle diameter of 2.25 inches, and angled 15 degrees above the horizontal plane where each nozzle provides a  $U_oD$  of 5.1 ft<sup>2</sup>/s. Per the TTR, there might be four abandoned and/or failed slurry pumps in Tank 42, two standard slurry pumps (Risers V1 and V2) and two quad pumps (Risers G and H).

The impact of the abandoned and/or failed slurry pumps in Tank 42 was assessed by performing a momentum balance over the contents of tank. The momentum of one of the jets discharging into the tank from the submersible blend pump is described by equation [12].<sup>4</sup>

$$\hat{m} = \rho U_o^2 \frac{\pi D^2}{4} = \rho \frac{\pi}{4} U_o^2 D^2 = \frac{\pi}{4} \frac{1.25 \text{ g}}{\text{cm}^3} \left( \frac{4738 \text{ cm}^2}{\text{s}} \right)^2 = 22,000,000 \frac{\text{g} \cdot \text{cm}}{\text{s}^2} \quad [12]$$

Multiplying the momentum flux by 2 to account for two nozzles produces the total momentum input into the tank, which is 44,000,000 g cm/s<sup>2</sup>.

The presence of the abandoned slurry pumps will cause drag as the fluid moves around them, which will reduce the momentum of the fluid in the tank. The pumps will be treated as cylinders and equation [13] can be used to calculate the drag caused by the abandoned slurry pumps

$$F_d = \frac{C_D \rho U^2 D_p L_p}{2} \quad [13]$$

where  $C_D$  is the drag coefficient,  $U$  is the velocity of the fluid at the abandoned slurry pump,  $D_p$  is the diameter of the abandoned slurry pump assembly (16 inches<sup>11</sup>), and  $L_p$  is the length of the pump assembly (348.28 inches).<sup>4</sup> A length of 348.28 inches (885 cm) was selected for conservatism, given the abandoned

pumps do not extend to the tank bottom. Because the blend pump nozzles are angled upward, the impact of the abandoned slurry pumps below the elevation of the nozzles should not be significant, but it is included for conservatism. The velocity of the fluid approaching the abandoned slurry pump can be estimated using CFD modeling performed as part of the pilot-scale testing conducted to select design parameters for the submersible blend pump.<sup>9</sup> Reviewing the CFD calculated velocity distribution in the horizontal plane of the discharge jet for a case (Case 11a and Figure 10 in the document) with a V-shape pump discharge, a  $U_0D$  of 0.58 ft/s at pilot-scale (6.3 ft/s at full-scale), and no cooling coils, and considering the location of the abandoned pumps in Tank 42 (see Attachment 2), the maximum fluid velocity in the region of the abandoned pump in Riser G is approximately 0.2 ft/s (6.1 cm/s), and the maximum velocity in the regions of the abandoned pumps in Risers H, V1, and V2 is less than 0.13 ft/s. Since the equivalent  $U_0D$  in this case is larger than the planned  $U_0D$  in Tank 42, the actual velocity in Tank 42 should be less. Since the CFD case had no cooling coils, the velocity in the region of the abandoned pumps in Tank 42 (which has cooling coils) is likely less. Because the pump discharge nozzles are angled 15° upward, the velocity may be higher above the pump nozzle horizontal plane and lower below the pump nozzle horizontal plane. Because the abandoned pumps are not in the direct path of the discharge jets from the blend pump, this effect is likely small. Given the other conservatisms discussed above, the estimates for the velocity in the region of the abandoned pumps are believed to be reasonable for this calculation.

The drag coefficient is a function of Reynolds number which is defined by equation [14]

$$Re = \frac{UD_p}{\nu} = \frac{6.1 \frac{\text{cm}}{\text{s}} 40.6 \text{cm}}{0.024 \frac{\text{cm}^2}{\text{s}}} = 10,319 \quad [14]$$

where  $D_p = 40.6$  cm, and  $\nu$  (the kinematic viscosity of the salt solution) =  $0.024 \text{ cm}^2/\text{s}$ .<sup>a</sup> For a velocity of 0.2 ft/s, the Reynolds number is 10,319 as shown in equation [14]. For a fluid velocity of 0.13 ft/s, the Reynolds number is 6,707. According to reference 4, at a Reynolds number of 1,000 – 100,000 for a cylinder, the drag coefficient is approximately 1. For the pump in Riser G, the drag force is calculated with equation [15].

$$F_d = 1 \frac{1.25 \text{g}}{\text{cm}^3} \left( \frac{6.1 \text{cm}}{\text{s}} \right)^2 \frac{40.6 \text{cm} 885 \text{cm}}{2} = 836,000 \frac{\text{g cm}}{\text{s}^2} \quad [15]$$

The drag force for each pump in Risers H, V1, and V2 is calculated with equation [16]

$$F_d = 1 \frac{1.25 \text{g}}{\text{cm}^3} \left( \frac{3.96 \text{cm}}{\text{s}} \right)^2 \frac{40.6 \text{cm} 885 \text{cm}}{2} = 352,000 \frac{\text{g cm}}{\text{s}^2} \quad [16]$$

Adding the drag forces from each of the abandoned slurry pumps, the total drag force is 1,892,000 gcm/s<sup>2</sup> or 4.3% of the momentum produced by the submersible blend pump. The abandoned pumps reducing fluid momentum in the tank is equivalent to the input momentum being reduced by 4.3%. Since the momentum is proportional to the square of  $U_0D$ , reducing  $U_0D$  by the square root of 1.043 would produce the same effect as the drag from the abandoned pumps. The square root of 1.043 is 1.021. Since the blend time is inversely proportional to  $U_0D$  (see equations [5], [6], and [9]), the impact of the abandoned pumps will be to increase the blend time by a maximum of 2.1%. Equation [7] and [8] result in a smaller increase in blend time, given the 2/3 power on  $U_0D$ . The actual loss of momentum and increase in blend time is likely less, because the cooling coils will reduce fluid velocity near the abandoned pumps, and the fluid velocity in the

<sup>a</sup> A kinematic viscosity of  $0.024 \text{ cm}^2/\text{s}$  was selected for the calculation of the Reynolds number based upon a viscosity of 3 cp and a fluid density of  $1.25 \text{ g/cm}^3$ . The viscosity is typical of 5.6 – 6.0 M sodium salt solution, and the density is typical of a 5.6 M sodium salt solution. The drag coefficient is not sensitive to changes in Reynolds number between 1,000 – 100,000, so any uncertainty in the viscosity or density has little impact on  $C_D$ .

bottom section of the tank will be less than the fluid velocity in the plane of the pump discharge jets. While the fluid velocity in the vicinity of the abandoned pumps was estimated based on CFD calculations in the horizontal plane with angled discharge jets, the uncertainty from this difference is likely small and accounted for with the conservatism. This is an additive loss to that of level changes described previously.

### 2.3 Quality Assurance

Requirements for performing reviews of technical reports and the extent of review are established in manual E7 2.60. SRNL documents the extent and type of review using the SRNL Technical Report Design Checklist contained in WSRC-IM-2002-00011, Rev. 2.

The work scope requested by SRR is describing in a Task Technical Request.<sup>2</sup> SRNL approach to performing the analysis, along with the quality assurance requirements, are described in the Technical Task and Quality Assurance Plan.<sup>12</sup>

## 3.0 Discussion

This analysis employed two approaches to evaluate the impact of changing the distance between the pump discharge nozzle and the liquid surface or the distance between the pump discharge nozzle and the sludge surface in the blend tanks that prepare feed for the SWPF. The approaches were matching the Froude number and examining existing jet mixing correlations.

This analysis predicted an increased blend time from increasing the distance between the pump discharge nozzle and the liquid surface, with the increase ranging from 4 - 9%, and an increased blend time from changing the distance between the pump elevation and the sludge surface, with the increase ranging from 15 - 33%. The fact that two very different approaches produce similar results and that multiple jet blending correlations predict similar results should increase the confidence in these results and support the conclusion that the impact of changing the distance between the pump discharge nozzle and the liquid surface or the distance between the pump discharge nozzle and the sludge surface should be small.

Decreasing the distance between the pump discharge nozzle and the liquid surface should not increase the blend time unless the level is reduced to the point that the jet produced by the blend pumps reaches the liquid surface before it reaches the wall. Decreasing the distance between the pump discharge nozzle and the sludge surface is unlikely to increase the blend time in these tanks, but it could lead to additional sludge suspension. SRR plans to maintain the minimum distance between the pump discharge nozzle and the sludge at 165.86 inches.

The analysis showed that the abandoned, failed slurry pumps will lead to a decrease in fluid velocity in the blend tanks. However, the abandoned pumps occupy a very small fraction of the tank, much of the fluid motion in the tank will not be impacted by these abandoned pumps, and the increase in blend time is predicted to be 2.1%.

## 4.0 Conclusions

The conclusions from this study follow.

- Increasing the liquid height in the blend tanks relative to the pump discharge nozzles could increase the blend time. The maximum increase is calculated to be 4 - 9% above the baseline.
- Increasing the pump discharge nozzle elevation relative to the sludge level (or tank bottom) in the blend tanks could increase the blend time. The maximum increase is calculated to be 15 - 33% above the baseline.
- Decreasing the distance between the pump discharge nozzle and the liquid surface to as little as 100 inches is unlikely to adversely impact the liquid blend time, but it could reduce it.

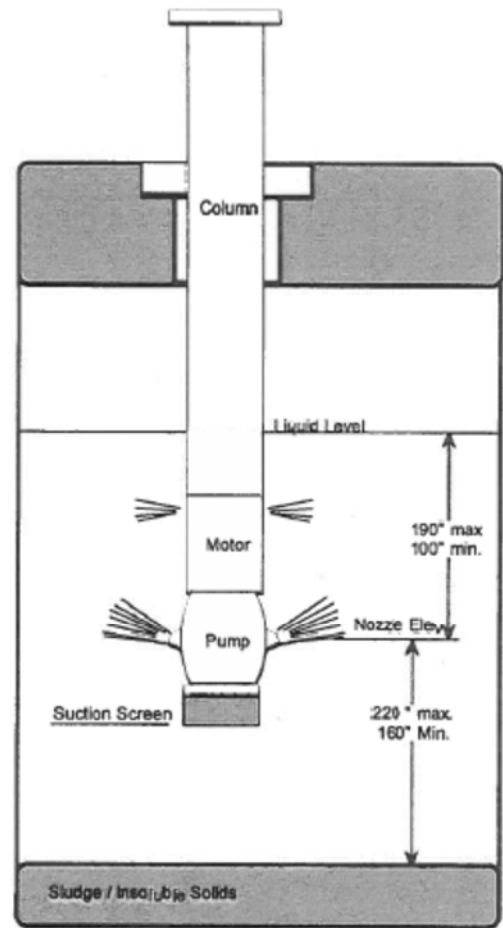
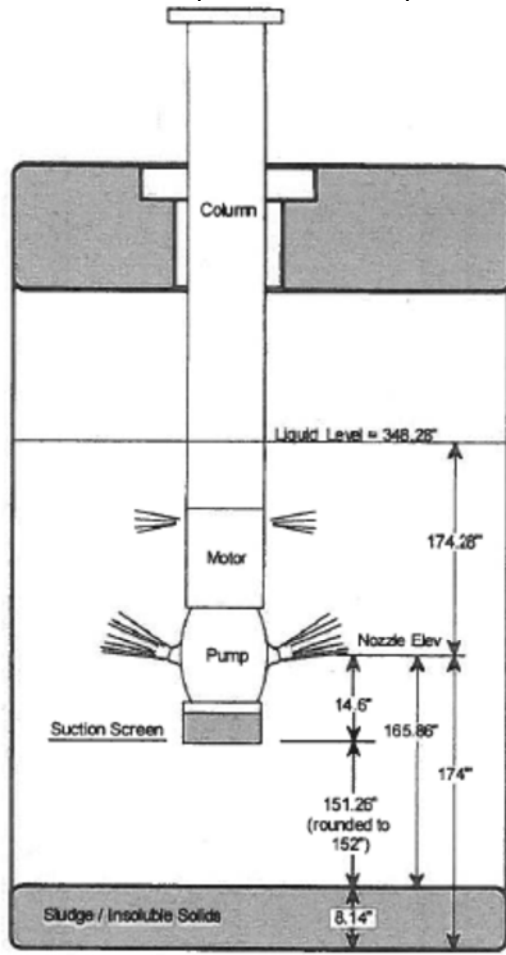
- The abandoned pumps in Tank 42 occupy a very small fraction of the tank, much of the fluid motion in the tank will not be impacted by these abandoned pumps, and the increase in blend time is predicted to be 2.1% above the baseline.

## 5.0 References

- 
- <sup>1</sup> R. A. Leishear, M. R. Poirier, and M. D. Fowley, “Blending Study for SRR Salt Disposition Integration: Tank 50H Scale Modeling and Computer Modeling for Blending Pump Design, Phase 2”, SRNL-STI-2011-00151, May 2011.
  - <sup>2</sup> G. Clendenon, “Impact of Blend Pump Installation Elevations on Mixing Effectiveness”, U-TTR-H-00046, January 24, 2019.
  - <sup>3</sup> T. M. Punch, “Submersible Blend Pump Operational Constraints to Prevent Solids Suspension”, U-ESR-G-00030.
  - <sup>4</sup> C. O. Bennett and J. E. Myers, Momentum, Heat, and Mass Transfer, 3<sup>rd</sup> Ed., New York: McGraw-Hill, 1982.
  - <sup>5</sup> R. K. Grenville and A. W. Nienow, “Blending of Miscible Liquids”, in E. L. Paul, V. A. Atiemo-Obeng, and S. M. Kresta, eds., Handbook of Industrial Mixing: Science and Practice, Hoboken: Wiley, 2004.
  - <sup>6</sup> B. K. Revill, “Jet Mixing”, in N. Harnby, M. F. Edwards, and A. W. Nienow, Mixing in the Process Industries, 2<sup>nd</sup> Ed., Boston: Butterworth-Heinemann, 1992.
  - <sup>7</sup> K. L. Wasewar, “A Design of Jet Mixed Tank”, Chem Biochem Eng, vol. 20, 2006, pp. 31-46.
  - <sup>8</sup> R. K. Grenville and J. N. Tilton, “Jet Mixing in Tall Tanks: Comparison of Methods for Predicting Blending”, Chem Eng Res Des, vol 89, 2011, pp. 2501-2506.
  - <sup>9</sup> S. Y. Lee and B. W. Armstrong, “SDI CFD Modeling Analysis”, SRNL-STI-2011-00025, April 2011.
  - <sup>10</sup> “Savannah River Plant Bldg 241-15H Tanks 38, 39, 40, 41, 42, &43 Additional waste Storage Tanks Top Slab Plan – Tank 42 Concrete and Steel, Drawing W700761.
  - <sup>11</sup> DPSTSA-200-10, SUP-18, Rev. 1.
  - <sup>12</sup> M. R. Poirier, “Task Technical and Quality Assurance Plan for Submersible Blend Pump Mixing Evaluation”, SRNL-RP-2019-00121, March 25, 2019.



Attachment 1. Liquid Level and Pump Elevation in SWPF Blend Tanks



Attachment 2. Layout of Tank 42

