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## REPAIR OF A MELTER POUR SPOUT USING AN EXPANDING RING<sup>†</sup>

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### ABSTRACT

An expanding ring was developed that provided remote repair of the pour spout for a radioactive waste vitrification melter. This passive device used gas pressure to expand a metal ring against the irregularly shaped pour spout wall. Laboratory modeling and testing were performed for proof of concept and optimization before final design and field deployment. The ring expanded radially more than 4.8 mm and successfully repaired the melter pour spout allowing continued glass pouring operation.

### INTRODUCTION

The Defense Waste Processing Facility (DWPF) vitrifies high level radioactive waste (HLW) from waste tanks at Savannah River Site (SRS). The process produces a suitable waste form for long-term dry storage of the HLW. Initially the chemistry of the liquid HLW is adjusted using various acids. Borosilicate glass frit is added, the slurry is condensed, and then it is fed into a high temperature glass melter. When melted the molten glass is poured into stainless steel canisters. Over the years, flow of molten glass down the pour spout has corroded the glass disengagement point to the extent that the stream could no longer be directed into the stainless steel storage canister without plugging.

The melter is operated within facility referred to as a canyon. The facility incorporates 1.4 m (4.5 ft) thick concrete walls to shield personnel from the radioactive waste. A significant engineering challenge exists for any modification or repair because all operations must be performed remotely. To repair the pour spout a simple and novel technique was developed that allows a circumferential seal against the pour spout wall. The expanding ring will seal against an irregular shaped surface. This pressurized device is remotely deployable requiring no complicated mechanical tooling to install. The ring is positioned into the heated section of the pour spout and allowed to expand. This paper describes the expanding ring developed for the melter pour spout repair.

<sup>†</sup> Patent Pending

### BACKGROUND

Approximately 136 million liters (36 million gallons) of highly radioactive waste solutions from the production of nuclear materials at the United States Department of Energy's Savannah River Site (SRS) are presently stored in large underground carbon steel tanks. SRS is closing the tank farm and dispositioning the inventory of high level waste. Vitrification technology was developed on site for immobilizing waste to allow for controlled decay of long-lived radionuclides and will be used to treat the waste solution.

Liquid high level waste sludge is transferred from waste tanks to the Defense Waste Processing Facility (DWPF). First the chemistry is adjusted by addition of nitric and formic acids. Borosilicate frit is then added and the slurry is concentrated until a solids content of 45 to 50 wt% is attained. The slurry is finally fed to a melter where it is melted. The resultant high level waste glass is poured into stainless steel canisters, which is part of the engineering package that will be used for long-term storage of the HLW.

DWPF's first glass melter began operation in May 1994 and was shut down for replacement in November 2002. It operated continuously for over eight years, including 2 years of non-radioactive cold chemical operations followed by 6 years of radioactive waste processing. Over 1400 canisters,  $2.4 \times 10^6$  kg (5.2 million pounds) of glass, have been successfully poured. This represents about 27 percent of the total glass to be produced in this facility.

The DWPF Melter (Figure 1) is a refractory lined cylindrical vessel. Heat is provided via Joule heating of the glass and by resistance heaters located in the melter plenum, riser, pour spout, and drain valve. Internal dimensions are approximately 1.83 m (6 ft) diameter by 2.2 m (7 ft) high. Nominal glass depth is 86 cm (34 in). Glass contact material is Monofrax™ K-3, a fused-cast chromia-alumina refractory. Vapor space refractory is Korundal™ XD. Metal components in contact with the molten glass and vapor space were fabricated from a nickel-base alloy, Inconel™ 690. These include thermowells, a bubbler, off gas film coolers, lid heaters, borescopes, electrodes, and the pour spout.

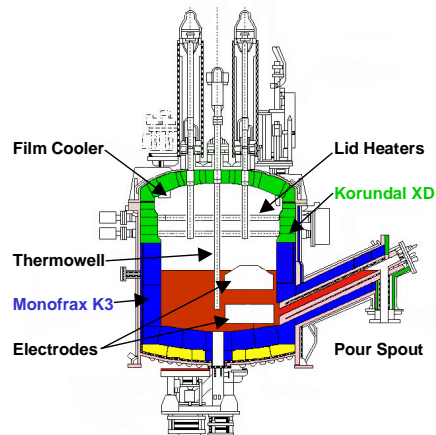


Figure 1. Schematic of the DWPF melter.

The melter is operated at a glass temperature of approximately 1150 °C. Once the slurry containing the borosilicate frit is melted the molten glass flows up the melter riser where it then flows down the pour spout into a stainless steel waste canister below. Two machined circumferential steps (upper and lower knife edges) in the pour spout are intended to allow the molten glass to disengage the pour spout and fall freely into the center of the canister (Figure 2). The knife edges, located at the 5 to 7.6 cm (2 to 3 inch) transition and the 7.6 to 10 cm (3 to 4 inch) transition, are maintained at 1100 °C with external heaters. The lower most 10 cm (4 inch) section has a temperature gradient from 1100 °C at the lower knife edge to approximately 500 °C at the bottom. Molten glass wets only about 30 % of the circumference of the pour spout. When the pour stream detaches from the upper knife edge it must fall through a non-heated portion of the pour spout before reaching the canister.

After approximately 3 years of operation, glass pouring problems led to remote visual inspection of the melter pour spout. This was accomplished by the use of a high temperature remotely operated borescope. Examination revealed significant material loss on the glass contact side of the pour spout (Figure 3). This figure represents a 360 degree panoramic view of the pour spout bore, showing the extent of material loss, estimated to be 6.4 mm (0.25 in) at the upper knife edge. This condition resulted in a significant reduction in melter throughput due to the frequent need for cleaning glass from the lower spout. The need to reestablish desired glass flow characteristics led to the development of remote cleaning tools, the nickel 200 bellows liner, and the replaceable pour spout insert. The nickel liner was installed because it shed molten glass better than the 304L liner<sup>(1)</sup>. The insert acted as a funnel by collecting the molten glass and allowing it to fall into the center of the canister once more. It also protected the 3 inch bore from further degradation. Subsequent metallurgical examination of a degraded insert established a nominal corrosion rate 4.78 mm/yr (0.188 in/yr) and the mechanism for the observed attack<sup>(2)</sup>.

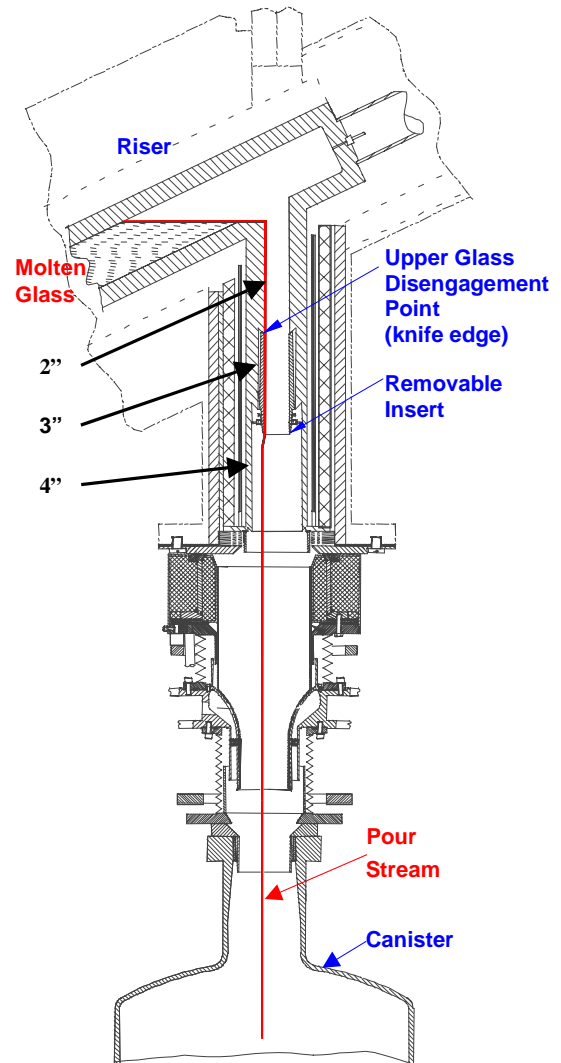


Figure 2. Melter pour spout showing the upper glass disengagement point.

Approximately seven years later the removable insert would no longer redirect the glass as desired and melter throughput suffered. Inspection of the upper portion of the 3 inch diameter bore showed that it had become elliptical with a major diameter of approximately 8.25 cm (3.25 inches). The

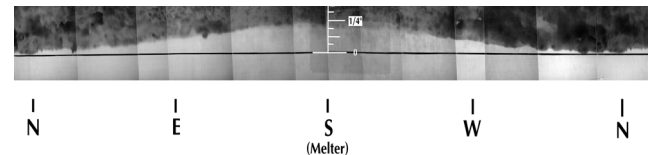


Figure 3. 360 degree panoramic view of the melter pour spout showing metal loss at the upper knife edge (S).

metal surface was also extremely rough. The observed degradation resulted because the 5 cm (2 inch) bore continued to experience material loss in the glass contact region. Thus the insert would no longer seal underneath the upper knife edge (Figure 2). The pour stream split with some going

behind the insert. The resulted in further corrosion of the pour spout wall. The portion of the pour stream that remained attached to pour spout wall solidified in the lower unheated section and eventually blocked glass flow into the canister.

Several unsuccessful attempts were made using special inserts and platinum gaskets to seal against the upper knife edge. Successful repair of the pour spout was only accomplished by using an expanding ring to produce an active, circumferential seal against the wall.

### EXPANDING RING DEVELOPMENT

The following design requirements were imposed for development and for installation of the expanding ring into the fully operational DWPF melter pour spout.

- 1) The ring had fit into the 7.6 cm (3 inch) bore and have a 5 cm (2 inch) inner diameter. It had to expand at least a 0.32 cm (0.125 inch) radially with the capability to expand an additional 0.76 mm (0.030 inch) into a contoured gouge which was designed to represent the degradation on the glass contact side of the pour spout.
- 2) Provide sufficient force necessary to make a permanent glass tight seal but not deform the pour spout wall.
- 3) Time to insert and position ring into the pour spout before expanding must be at least three minutes.
- 4) All materials must be compatible with existing materials and the corrosive environment (glass and vapor).

The expanding ring consists of an inner and outer ring as shown in Figure 4. The rings were machined so there is a

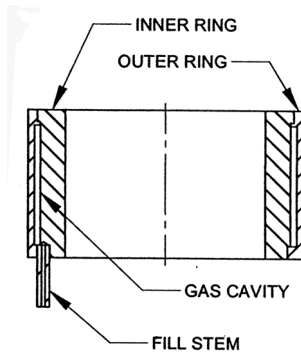


Figure 4. Expanding Ring Concept

small internal cavity. The inner and outer rings are electron-beam welded to provide a full penetration weld. The internal cavity is purged of residual moisture and then filled with pressurized helium gas through the attached fill stem. After pressurizing the fill stem is pinch welded to seal the gas inside the chamber. Helium was chosen as the fill gas primarily because it made leak detection after fabrication easy. The diffusivity of helium gas in face-centered-cubic metals at high temperature is low; therefore, a positive pressure would be maintained for a longer period of time.

The wall thickness of the inner and outer ring, for this application, is designed so the stress in the outer ring is larger in order to cause deformation or bulging of the surface. Permanent deflection occurs when the stress due to the

internal pressure exceeds the material yield stress at temperature.

The gas pressure is calculated using the Ideal Gas Law, which is a function of cavity volume and temperature. The relationship between pressure, volume and temperature is given by:

$$\frac{P_1 V_1}{T_1} = \frac{P_2 V_2}{T_2}$$

where subscript 1 refers to initial conditions and subscript 2 refers to conditions at temperature.

### FINITE ELEMENT MODELING

Finite element modeling was used to aid ring design by investigating the deformation characteristic as a function of increasing temperature and by determining the stress distribution in the ring and pour spout wall. Axisymmetric models of the ring and the DWPF pour spout were constructed and analyzed using the ABAQUS computer code<sup>(3)</sup>. The temperature dependent stress-strain characteristics of Inconel 690 were obtained from the literature<sup>(4,5)</sup>. The material was considered nearly perfectly plastic at 1050 °C.

As the ring temperature increases the gas pressure increases until the material reaches its yield strength. The outer wall of the ring then expands until the pressure is no longer sufficient to cause yielding of additional material. The final stress distribution from a typical cross-section from the model of an expanding ring against the pour spout wall is shown in Figure 5. The initial gas fill pressure was 2.76 kPa

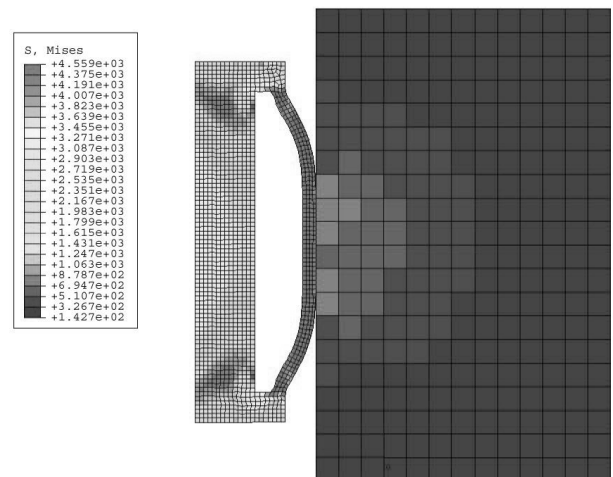


Figure 5. Mises stress results from the finite element analysis of an axisymmetric Inconel 690 model with an initial 3.2 mm (1/8 inch) radial clearance between the ring and wall.

(400 psig) at room temperature, and the initial gas volume was 10.3 cm<sup>3</sup> (0.63 in<sup>3</sup>).

For the 1.6 mm (1/16 inch) outer wall, the ring yields at a calculated Mises stress of about 31.7 kPa (4600 psi). The maximum stress level for the 3/8 inch inner ring wall was calculated to be less than 20.7 kPa (3000 psi). When fully

expanded, the stress on the pour spout wall was approximately 6.9 kPa (1000 psi) at the contact area.

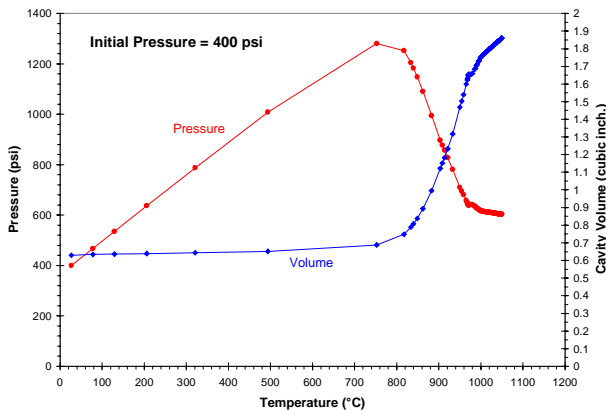


Figure 6. Calculated pressure and cavity volume changes as a function of temperature for the expanding ring.

Ring pressure and expansion as a function of temperature were calculated using ABAQUS and are shown in Figure 6. Internal gas pressure increases with temperature until the material yields at approximately 800 °C. At this temperature the cavity begins to expand. Above 800 °C, the pressure decreases rapidly as the cavity becomes larger. At approximately 960 °C, the outer surface reaches the wall. With increasing temperature, the cavity continues to expand, flattening against the wall. Finally, the membrane stress becomes less than the yield strength and no further expansion occurs.

### EXPERIMENTAL TESTING

The expanding ring was developed and tested in a massive Glo-Bar™ furnace in the Materials Laboratory prior to deployment in the field. An Inconel 690 ring and pour spout assembly was tested with a maximum radial clearance of 0.4763 cm (0.1875 inch). An additional 0.76 mm (0.030 inch) was machined in a region of the pour spout mockup of about 30 degrees around the circumference to simulate the contour of the degraded pour spout. No attempt was made to duplicate surface roughness in laboratory tests. The simulated pour spout was coated with molten glass prior to ring insertion to approximate field conditions.

An assembled ring used in the testing program is shown in Figure 7. Type K thermocouples were attached to the ring to

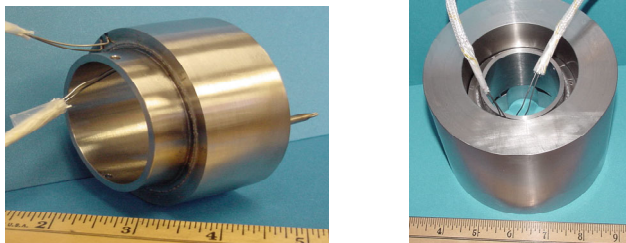


Figure 7. Pressurized Inconel 690 expanding ring with fill stem and attached thermocouples (left). Ring inserted into simulated pour spout (right).

measure temperature, which was recorded by a data acquisition system at 1 second intervals.

### TEST RESULTS

A graph of displacement versus time for an actual test and that predicted by the finite element model is shown in Figure

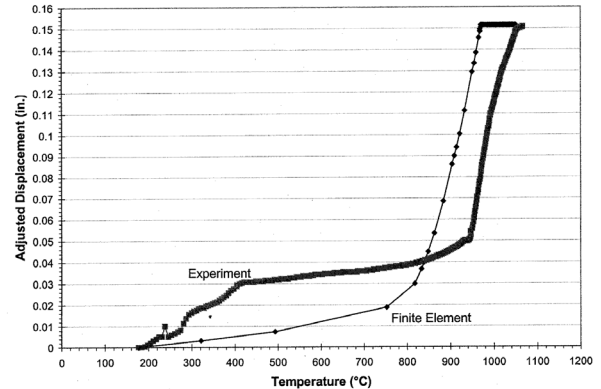


Figure 8. Experimental and calculated ring expansion for 1/8 inch radial gap including 0.03 inch contoured groove.

8. Although there was a slight difference initially between the measured and predicted data, which was attributed to the measuring system, finite element modeling was useful for predicting expansion rates of modified designs.

Multiple tests were performed to optimize radial expansion, axial contact, expansion rates, and the force exerted on pour spout wall. In addition, sealing capability and removal force for both a pressurized ring and a ring that was installed but depressurized were evaluated. The maximum radial expansion tested was 0.6 cm (0.218 inches) with an axial contact of 2 cm (0.75 inches). Time to begin expansion ranged from 3 to 6 minutes depending on the design. Static loads up to 32 kg (70 lbs) were required to initiate ring movement. A significantly higher load, 154 kg (340 lbs), was required to start movement after extended elevated temperature exposure. To test the seal additional glass frit was placed on top of the ring after it had been expanded. The assembly was allowed to soak in the furnace for several more hours before evaluation. Molten glass did not seep past the seal even in the contoured gouge (Figures 9 and 10). It was

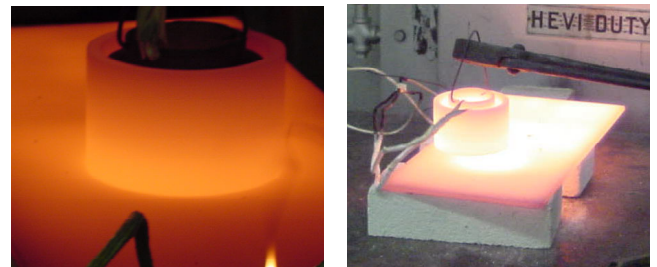


Figure 9. Photographs of expanding ring after being installed in simulated pour spout (left) and after removal from furnace (right).



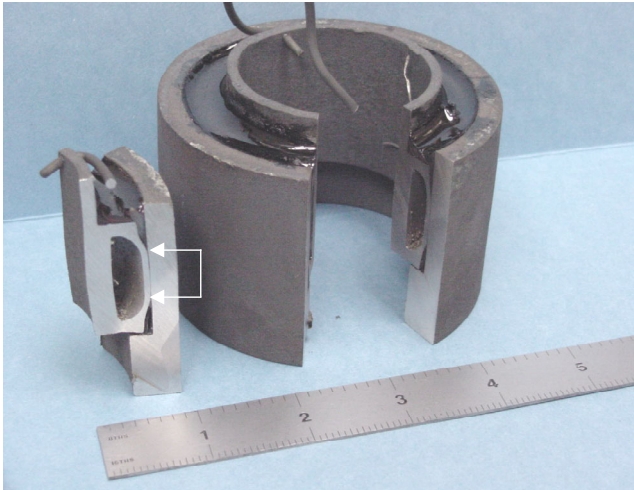


Figure 10. Destructive evaluation of ring/simulated pour spout assembly with 0.159 cm (1/16 inch) radial clearance after static molten glass exposure. Total axial contact was 2 cm (0.75 inches) (see arrow).

found that even a depressurized ring would still seal effectively and required significant force to initiate movement.

#### FIELD DEPLOYMENT

After proof of principal testing was completed the expanding ring was deployed to the field. Special tooling was developed to remotely position the expanding ring into the melter pour spout. This was accomplished by using a remote telerobotic arm (TRM) equipped with video cameras that could reach 3.7 m (12 feet) to the melter pour spout. Installation went as planned with insertion and positioning at the upper knife edge taking less than a minute. The expanding ring was allowed to soak for at least ½ hour at 1100 °C to ensure an adequate circumferential seal had been obtained

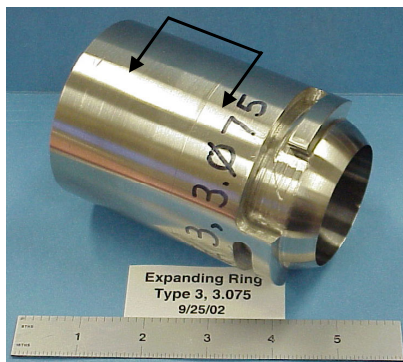


Figure 11. Photograph of completed expanding ring assembly including lower ball joint for mating insert. Expanding ring portion is shown between arrows.

before initiating glass pouring. The pour stream was stable and directed the molten glass into the canister. There was no indication of significant glass flow behind the expanding ring and insert. Figure 11 shows fully fabricated and charged expanding ring assembly prior to deployment to the field.

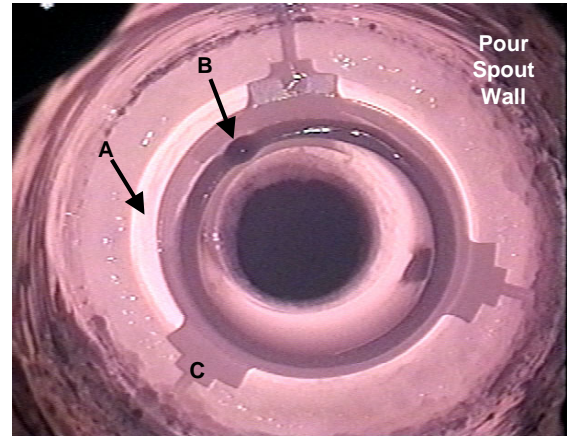


Figure 12. Remote video image taken off-center of the DWPF melter pour spout showing the expanding ring and insert installed after several glass pours. A) Expanded ring, B) Mating removable insert with residual glass at disengagement point, C) Insert retaining pins.

Figure 12 shows a photograph of the ring installed in the pour spout during an inspection after glass pouring operations had resumed.

#### CONCLUSIONS

Based on the results of the laboratory testing and field implementation the conclusions are:

- 1) The expanding ring successfully created a glass tight circumferential seal against the irregular surface of the DWPF melter pour spout.
- 2) The expanding ring concept is simple because it only requires temperature for activation and does not require complicated mechanical tooling to install.
- 3) Radial expansions of up to 4.8 mm (3/16 inch) were obtained with the currently designed expanding ring. Larger expansions may be possible.
- 4) The expanding ring supported static loads up to 154 kg (340 lbs) in laboratory testing using machined surfaces. Higher loads may be required before initiating ring movement on the roughened surface of the melter pour spout.

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