

This document was prepared in conjunction with work accomplished under Contract No. DE-AC09-76SR00001 with the U.S. Department of Energy.

DISCLAIMER

This report was prepared as an account of work sponsored by an agency of the United States Government. Neither the United States Government nor any agency thereof, nor any of their employees, makes any warranty, express or implied, or assumes any legal liability or responsibility for the accuracy, completeness, or usefulness of any information, apparatus, product or process disclosed, or represents that its use would not infringe privately owned rights. Reference herein to any specific commercial product, process or service by trade name, trademark, manufacturer, or otherwise does not necessarily constitute or imply its endorsement, recommendation, or favoring by the United States Government or any agency thereof. The views and opinions of authors expressed herein do not necessarily state or reflect those of the United States Government or any agency thereof.

This report has been reproduced directly from the best available copy.

Available for sale to the public, in paper, from: U.S. Department of Commerce, National Technical Information Service, 5285 Port Royal Road, Springfield, VA 22161, phone: (800) 553-6847, fax: (703) 605-6900, email: orders@ntis.fedworld.gov online ordering: <http://www.ntis.gov/ordering.htm>

Available electronically at <http://www.doe.gov/bridge>

Available for a processing fee to U.S. Department of Energy and its contractors, in paper, from: U.S. Department of Energy, Office of Scientific and Technical Information, P.O. Box 62, Oak Ridge, TN 37831-0062, phone: (865) 576-8401, fax: (865) 576-5728, email: reports@adonis.osti.gov

SAVANNAH RIVER LABORATORY
TECHNICAL DIVISION

DPST-83-828

Acc. No. 106396

CC: E. B. Warming, L11W09
W. R. Stevens, III, 773-A
J. F. Ortaldo, 773-A
W. W. F. Yau, 709-G
M. D. Boersma
D. G. Kilpatrick
L. F. Landon
C. T. Randall
D. C. Iverson
T. L. Allen
SRL Records (4)

TO: D. C. WITT

TIS FILE
RECORD COPY

FROM: A. E. HAILEY *AEH*

September 15, 1983

PERFORMANCE CHARACTERISTICS OF AN ISOTHERMAL FREEZE VALVE

INTRODUCTION

A freeze valve has been specified for draining the DWPF melter at the end of its lifetime. Two freeze valve designs have been evaluated on the Small Cylindrical Melter-2 (SCM-2). In order to size the DWPF freeze valve, the basic principles governing freeze valve behavior need to be identified and understood.

The operating principles of a freeze valve involve forming a glass plug to stop flow and melting the plug to initiate flow. The plug is formed and maintained by cooling a section of pipe which runs from the melter floor to the outside bottom of the melter shell. Heaters are used to melt the plug and initiate glass flow. For the glass formations and the melt pool depth planned for use in the DWPF a method is required for stopping the glass flow long enough to permit the cooled section of pipe to freeze the plug. A jet controlled valve that uses air pressure to stop the glass flow, has been successfully demonstrated on the Small Cylindrical Melter-2.

This paper discusses the performance characteristics of a freeze valve once glass flow is initiated. This is essentially an isothermal process. A theoretical model of the process is developed, verified, and then used on a DWPF-scale melter to evaluate the effect of glass properties and valve geometry on freeze valve performance. A major portion of the theory was developed by K. R. Routt.

SUMMARY

The flow behavior of an isothermal freeze valve has been modeled. Glass flow through the valve is essentially a steady-state balance between the static head of the glass in the melter and the viscous forces generated along the walls of the valve. The model is limited to cases in which laminar flow is encountered. Calculations show that laminar flow occurs for all glasses with viscosities greater than 10 poise.

The model was verified by running freeze valve simulations, using silicone fluid in place of molten glass. Agreement between model predictions and experimental results was good.

The performance characteristics of a DWPF type freeze valve were also modeled, assuming a melt pool diameter of 6 feet, a glass depth of 2.8 feet and a glass viscosity of 60 poise. Drain rates were predicted (in pounds of glass per hour) for several valve sizes. A 1.0-inch-ID valve, 3 feet long would drain the melter at an average rate of 930 lb/hr (assuming glass melt rate viscosity of 80 poise). A valve this size could drain a completely full melter in about 15 hours (excluding time required for canister changes).

Theory

A typical freeze valve design is shown in Figure 1. Basically it consists of an inner pipe surrounded by an air cooled annulus and one of several types of heaters. During routine melter operation, the inner pipe is cooled by means of air flow in the surrounding annulus to some temperature below the softening point of the glass so that glass flow through the inner pipe does not occur. When it is desired to initiate glass flow, the air is turned off, and the glass in the inner pipe is heated by one of several techniques* until glass flow into the canister begins. When the canister is filled to the desired level, the glass flow must be stopped, and it is the technique for stopping the glass flow that is of primary interest in the discussion that follows.

* See DPST-81-961 for a description of possible heating techniques.

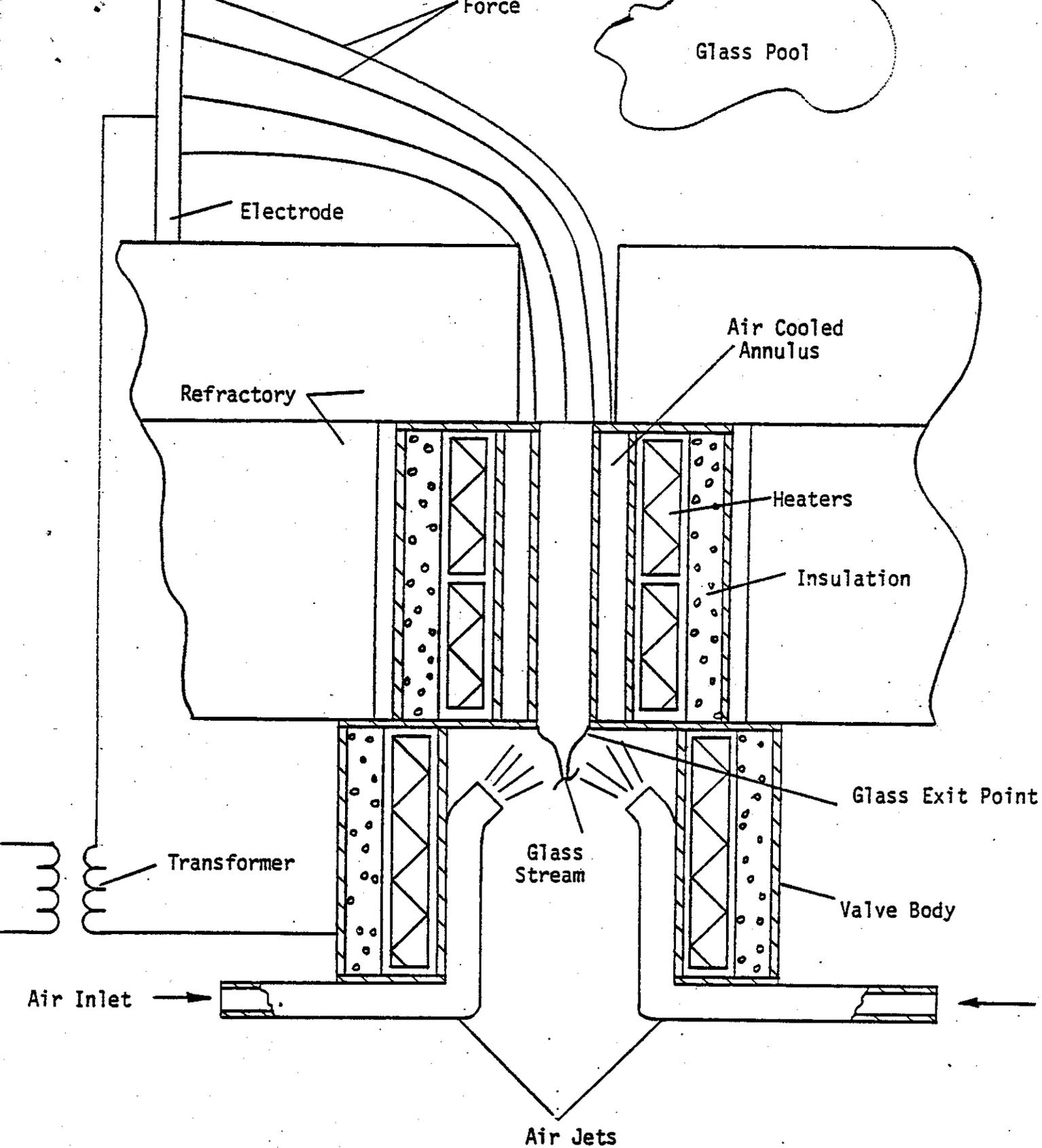


FIGURE 1 - JET CONTROLLED FREEZE VALVE

Conservation of mass in the melt pool above the freeze valve requires that

$$-\rho\pi R_{gp}^2 \frac{d}{dt} [z_{bgp} - z_{ms}(t)] = \rho\pi R_v^2 v(t) \quad (1-1)$$

$$z_{ms} < z_{bgp}$$

where

ρ = density of the glass [lb_m/ft^3],

R_{gp} = radius of the glass pool [ft],

z_{bgp} = distance from a reference elevation to the bottom of the glass pool [ft],

$z_{ms}(t)$ = distance from a reference elevation to the melt surface at time t [ft],

R_v = inside radius of the inner pipe of the freeze valve [ft],

$v(t)$ = glass velocity through the inner pipe of the freeze valve at time t [ft/sec],

t = time [sec].

Equation (1-1) simply relates the rate at which the melt surface falls to the rate at which glass flows from the melt pool into the top of the freeze valve.

Conservation of momentum in the melt pool above the freeze valve requires that

$$\begin{aligned} \rho\pi R_{gp}^2 \frac{d}{dt} \left\{ [z_{bgp} - z_{ms}(t)] \frac{dz_{ms}(t)}{dt} \right\} = & -\rho\pi R_v^2 v(t)^2 \\ + 144 g_o P_p \pi R_{gp}^2 - 144 g_o P_1(t) \pi (R_{gp}^2 - R_v^2) \\ - 144 g_o P_2(t) \pi R_v^2 + \rho\pi R_{gp}^2 [z_{bgp} - z_{ms}(t)] g \end{aligned} \quad (1-2)$$

$$z_{ms} < z_{bgp}$$

where

P_p = absolute pressure in the plenum [psia],

g_o = standard gravitational constant [32.14 lb_m-ft/sec²-lb_f],

P_1 = absolute pressure on the melter floor away from the freeze valve [psia],

P_2 = absolute pressure at the top of the freeze valve [psia],

g = local gravity [ft/sec²].

The left hand side of (1-2) represents the net change in momentum of the glass pool. The first term on the right hand side is the momentum lost from the glass pool due to glass flowing through the valve. The second, third, and fourth terms are the sum of the pressure forces acting on the top and bottom surfaces of the glass pool. The fifth term is the gravitational force acting on the glass pool.

If one requires that the amount of glass in the freeze valve be constant so that there is no net mass constant so that there is no net mass increase or decrease in the valve with respect to time,* and further assumes that the glass is incompressible (i.e., constant density), then conservation of mass for the glass in the freeze valve requires that the glass velocity v be independent of position in the valve. In other words, the glass velocity at the top of the valve is the same as the velocity at the bottom of the valve.

Conservation of momentum for the glass in the freeze valve requires that

$$\begin{aligned} \rho \pi R_v^2 L_v \frac{dv(t)}{dt} &= 144 g_o P_2(t) \pi R_v^2 \\ &+ \rho \pi R_v^2 L_v g \\ &- 8\pi \mu(\bar{T}) L_v v(t) \\ &- 144 g_o P_{bv} \pi R_v^2 \end{aligned} \quad (1-3)$$

* This is true as long as glass is flowing through the valve and the level z_{ms} does not drop below z_{bgp} . When $z_{ms} > z_{bgp}$ the melter is assumed to be virtually empty of glass.

where

L_v = length of inner pipe of freeze valve [ft],

g_o = standard gravitational constant [32.14 lb_m-ft/sec²-lb_f],

$\mu(\bar{T})$ = glass viscosity at the average glass temperature \bar{T} as it flows through the valve [lb_m/ft-sec],

P_{bv} = absolute pressure at bottom of the inner pipe of the freeze valve [psia].

The left hand side of (1-3) represents the net change of momentum of the glass flowing through the valve as the velocity changes with respect to time, i.e., the effect of inertia. The terms on the right hand side respectively represent the downward force at the top of the valve, the gravitational force acting on the glass in the valve, the frictional force between the flowing glass and the inner pipe wall for laminar flow and the upward force due to the pressure at the bottom of the inner pipe.

Equations (1-1), (1-2), and (1-3) are coupled, nonlinear, differential equations in the unknowns $z_{ms}(t)$, $v(t)$, $P_1(t)$, and $P_2(t)$. The necessary fourth equation can be obtained by approximating

$$144 g_o P_1(t) \cong [144 g_o P_p + \rho g (z_{bgp} - z_{ms}(t))]. \quad (1-4)$$

Dividing (1-2) through by πR_v^2 and using (1-4) and (1-3) to eliminate $P_1(t)$ and $P_2(t)$ gives

$$\begin{aligned} \frac{\rho R_{gp}^2}{R_v^2} \frac{d}{dt} \left\{ [z_{bgp} - z_{ms}(t)] \frac{dz_{ms}(t)}{dt} \right\} &= - \rho v(t)^2 \\ + \rho g [z_{bgp} - z_{ms}(t)] &+ 144 g_o (P_p - P_{bv}) \\ - \rho L_v \frac{dv(t)}{dt} + \rho L_v g &- \frac{8 \mu(\bar{T}) L_v v(t)}{R_v^2} \end{aligned} \quad (1-5)$$

Now (1-1) can be integrated to give the depth of the glass pool $D_{gp}(t)$ as

$$D_{gp}(t) = z_{bgp} - z_{ms}(t) = z_{zbp} - z_{ms}(o) - \frac{R_v^2}{R_{gp}^2} \int_0^t v(t) dt \quad (1-6)$$

Also (1-1) can be differentiated to give

$$\frac{d z_{ms}(t)}{dt} = \frac{R_v^2}{R_{gp}^2} v(t) \quad (1-7)$$

and

$$\frac{d^2 z_{ms}(t)}{dt^2} = \frac{R_v^2}{R_{gp}^2} \frac{dv(t)}{dt} \quad (1-8)$$

The derivative on the left hand side of (1-5) can be made a linear function in $v(t)$ by using

$$\begin{aligned} \frac{d}{dt} \left\{ \left[z_{bgp} - z_{ms}(t) \right] \frac{d z_{ms}(t)}{dt} \right\} &\equiv \frac{d}{dt} \left\{ \left[z_{bgp} - \bar{z}_{ms} \right] \frac{d z_{ms}(t)}{dt} \right\} \\ &\equiv \left[z_{bgp} - \bar{z}_{ms} \right] \frac{d^2 z_{ms}(t)}{dt^2} \\ &\equiv \left[z_{bgp} - \bar{z}_{ms} \right] \frac{R_v^2}{R_{gp}^2} \frac{dv(t)}{dt} \quad (1-9) \end{aligned}$$

where

$$\bar{z}_{ms} = \frac{\int_0^{t_d} z_{ms}(t) dt}{\int_0^{t_d} dt} \quad (1-10)$$

and t_d = time required to completely drain the glass pool [sec].

Using (1-7), the integral in (1-10) can be transformed into

$$z_{ms} = \frac{\int_{z_{ms}(0)}^{z_{ms}(t_d)} z_{ms}(t) d z_{ms}(t)}{\int_{z_{ms}(0)}^{z_{ms}(t_d)} d z_{ms}(t)}$$

$$\begin{aligned}
&= \frac{1}{2} \left[z_{ms}(t_d) - z_{ms}(0) \right] \\
&= \frac{1}{2} \left[z_{bgp} - z_{ms}(0) \right] \tag{1-11}
\end{aligned}$$

The nonlinear term in $v(t)^2$ on the right hand side of (1-5) can be made a linear term in $dv(t)/dt$ by expanding $v(t)$ in a Taylor's series to obtain

$$\begin{aligned}
v(t)^2 &= \left[v(t-\Delta t) + \Delta t \frac{dv(t)}{dt} + \dots \right]^2 \\
&\cong v(t-\Delta t)^2 + 2v(t-\Delta t) \Delta t \frac{dv(t)}{dt} \tag{1-12}
\end{aligned}$$

where

Δt = time step size [sec].

Using (1-6), (1-7), (1-10), (1-11), and (1-12), it is now possible to write (1-5) as a linear, integro-differential equation in $v(t)$, i.e.,

$$\begin{aligned}
& - \frac{\rho R_V^2}{R_{gp}^2} \left[v(t-\Delta t)^2 + 2v(t-\Delta t) \Delta t \frac{dv(t)}{dt} \right] \\
& + \rho \left[z_{bgp} - z_{ms}(t-\Delta t) - \frac{\Delta t R_V^2}{R_{gp}^2} v(t-\Delta t) \right] \frac{dv(t)}{dt} = \\
& - \rho v(t-\Delta t)^2 - 2\rho v(t-\Delta t) \Delta t \frac{dv(t)}{dt} + 144 g_o (P_p - P_{bv}) \\
& - \rho L_V \frac{dv(t)}{dt} \\
\rightarrow & \frac{-8 \mu(\bar{T}) L_V v(t)}{R_V^2} + \rho g \left[z_{bgp} - z_{ms}(0) - \frac{R_V^2}{R_{gp}^2} \int_0^t v(t) dt \right] + \rho g L_V
\end{aligned} \tag{1-13}$$

Now define

$$\alpha_1 = \frac{-2\rho R_v^2 v(t-\Delta t)\Delta t}{R_{gp}^2} + \rho \left[z_{bgp} - z_{ms}(t-\Delta t) - \frac{\Delta t R_v^2}{R_{gp}^2} v(t-\Delta t) \right] + 2\rho v(t-\Delta t)\Delta t + \rho L_v \quad (1-14)$$

$$\alpha_2 = \frac{8 \mu(\bar{T})L_v}{R_v^2} \quad (1-15)$$

$$\alpha_3 = \frac{\rho g R_v^2}{R_{gp}^2} \quad (1-16)$$

$$\alpha_4 = \rho v(t-\Delta t)^2 \left[\frac{R_v^2}{R_{gp}^2} - 1 \right] + \rho g \left[z_{bgp} - z_{ms}(o) \right] + \rho g L_v \quad (1-17)$$

so that (1-13) can be written as

$$\alpha_1 \frac{dv(t)}{dt} + \alpha_2 v(t) + \alpha_3 \int_0^t v(t)dt = \alpha_4 + 144 g_o (P_p - P_{bv}) \quad (1-18)$$

Equation (1-18) is the result of eliminating $P_1(t)$, $P_2(t)$, and $z_{ms}(t)$ in $v(t)$ results. It will now be solved for $v(t)$ using conventional techniques, after which $v(t)$ will then be used to express $P_1(t)$, $P_2(t)$, and $D_{gp}(t)$.

Uninterrupted Isothermal Glass Flow

Assume that at time $t=0$, the freeze valve opens and glass flow begins. No attempt is made to interrupt the flow. Instead the glass is allowed to flow freely from the glass pool until the entire contents of the melter have been drained. The pressure in the plenum P_p and at the bottom of the valve P_{bv} are held constant as the glass is drained.

In (1-18), assume a homogeneous solution of the form

$$v_h(t) = e^{mt} \quad (2-1)$$

which then yields

$$\alpha_1 m e^{mt} + \alpha_2 e^{mt} + \frac{\alpha_3}{m} e^{mt} = 0 \quad (2-2)$$

The characteristic equation of (2-2) is

$$\alpha_1 m^2 + \alpha_2 m + \alpha_3 = 0 \quad (2-3)$$

which has the roots

$$m_1 = \frac{-\alpha_2 + \sqrt{\alpha_2^2 - 4\alpha_1\alpha_3}}{2\alpha_1} \quad (2-4)$$

$$m_2 = \frac{-\alpha_2 - \sqrt{\alpha_2^2 - 4\alpha_1\alpha_3}}{2\alpha_1} \quad (2-5)$$

Thus the homogeneous solution of (1-18) is

$$v_h(t) = c_1 e^{m_1 t} + c_2 e^{m_2 t} \quad (2-6)$$

where c_1 and c_2 are constants to be determined.

For a particular solution v_p of (1-18) try

$$v_p = c_3 \quad (2-7)$$

which when substituted into (1-18) gives

$$\alpha_2 c_3 + \alpha_3 c_3 t = \alpha_4 + 144 g_0 (P_p - P_{bv}) \quad (2-8)$$

Since (2-8) is valid for all values of t , it must also be valid at $t=0$, which gives

$$c_3 = \frac{\alpha_4 + 144 g_0 (P_p - P_{bv})}{\alpha_2} \quad (2-9)$$

The general solution to (1-18) is the sum of the homogeneous and particular solutions, i.e.,

$$v(t) = c_1 e^{m_1 t} + c_2 e^{m_2 t} + c_3 \quad (2-10)$$

$$z_{ms} < z_{bgp}$$

To determine c_1 and c_2 , two boundary conditions are needed. If at time $t=0$, one required that $v(0) = 0$, then (2-10) yields

$$0 = c_1 + c_2 + c_3 \quad (2-11)$$

For the second boundary condition (1-3) may be used to give

$$\frac{dv(0)}{dt} = \frac{144 g_0}{\rho L_v} \left[P_2(0) - P_{bv} \right] + g \quad (2-12)$$

where

$$P_2(0) \equiv P_p + \frac{\rho g}{144 g_0} \left[z_{bgp} - z_{ms}(t) \right] \quad (2-13)$$

Using (2-12) in (2-10) gives

$$\frac{144 g_0}{\rho L_v} \left[P_2(0) - P_{bv} \right] + g = c_1 m_1 + c_2 m_2 \quad (2-14)$$

Equations (2-11) and (2-14) can be solved to yield

$$c_1 = \frac{1}{m_1 - m_2} \left\{ \frac{144 g_0}{\rho L_v} \left[P_2(0) - P_{bv} \right] + g + c_3 m_2 \right\} \quad (2-15)$$

$$c_2 = - (c_1 + c_3) \quad (2-16)$$

which completes the definition of the constants for $v(t)$ in (2-10).

Using (2-10) in (1-6) gives the depth of the glass pool at any time t as

$$D_{gp}(t) = z_{bgp} - z_{ms}(0) \frac{-R_v^2}{R_{gp}^2} \left[\frac{c_1}{m_1} \left(e^{m_1 t} - 1 \right) + \frac{c_2}{m_2} \left(e^{m_2 t} - 1 \right) + c_3 t \right] \quad (2-17)$$

$$z_{ms} < z_{bgp}$$

The pressure at the bottom of the melt pool away from the freeze valve can be obtained from (1-4), (1-6), and (2-17), i.e.,

$$P_1(t) \cong \frac{\rho g}{144 g_o} D_{gp}(t) \quad (2-18)$$

$$z_{ms} < z_{bgp}$$

Flow Interruption by Pressure Drop Control

One method of interruption the glass flow through the valve is by control of the applied pressure drop across the valve. This can be accomplished by using air jets directed at the bottom of the valve to increase the valve bottom pressure P_{bv} or alternatively, by causing an appropriate drop in the plenum pressure P_p . In this section, the response of the glass flow through the valve to a step change in the applied pressure drop across the valve will be studied.

Assume that glass is flowing isothermally through the valve and at some time t' there is a step change in the valve bottom pressure such that

$$\begin{aligned} P_{bv} &= P_{bv}, \quad t < t' \\ &= P_{bv}^*, \quad t > t' \end{aligned} \quad (3-1)$$

where

$$P_{bv}^* > P_{bv} \quad (3-2)$$

The solution to this problem is governed by the same equations that were derived and solved in sections 1 and 2 except that the boundary conditions are different. Equation (1-18) remains the same except that P_{bv} is replaced by P_{bv}^* . The characteristic Equation (2-2) and its roots (2-4) and (2-5) also remain the same. or a particular solution of the analog of (1-18), let

$$v_p = c_3^* \quad (3-3)$$

so that the analog of (2-9) becomes

$$c_3^* = \frac{\alpha_4 + 144 g_o (P_p - P_{bv}^*)}{\alpha_2} \quad (3-4)$$

let the analogous general solution (2-10) be

$$v^*(t) = c_1^* e^{m_1(t-t')} + c_2^* e^{m_2(t-t')} + c_3^* \quad (3-5)$$

$$t > t' \quad z_{ms} < z_{bgp}$$

To determine c_1^* and c_2^* , require that the velocity given by the two solutions be continuous at time t

$$v^*(t') = v(t') \quad (3-6)$$

where the right hand side of (3-6) is given by (2-10) evaluated at time t' using (3-6) in (3-5) gives

$$v(t') = c_1^* + c_2^* + c_3^* \quad (3-7)$$

For a second boundary condition, require that the slope of the velocity (i.e., the acceleration) given by the two solutions be continuous at time t' . Equating the derivatives of (2-10) and (3-5) at time t' gives

$$c_1 m_1 e^{m_1 t'} + c_2 m_2 e^{m_2 t'} = c_1^* m_1 + c_2^* m_2 \quad (3-8)$$

Equation (3-7) and (3-8) can be solved simultaneously for c_1^* to yield

$$c_1^* = \frac{c_1 m_1 e^{m_1 t'} + c_2 m_2 e^{m_2 t'} + m_2 [c_3^* - v(t')]}{m_1 - m_2} \quad (3-9)$$

$$c_2^* = v(t') - c_1^* - c_3^* \quad (3-10)$$

where c_1 and c_2 are defined by (2-15) and (2-16), and $v(t')$ is the solution to (2-10) at time t' . Equations (2-4) and (2-5) define m_1 and m_2 .

For any time $t > t'$, the depth of the glass pool can be written from (1-6) as

$$D_{gp}^*(t) = z_{bgp} - z_{ms}(o) \frac{-R_v^2}{R_{gp}^2} \left[\int_0^{t'} v(t) dt + \int_{t'}^t v^*(t) dt \right] \quad t > t' \quad (3-11)$$

However, by inspection of (1-6), (3-11) can be simplified to

$$D_{gp}^*(t) = D_{gp}^*(t') - \frac{R_v^2}{R_{gp}^2} \int_{t'}^t v^*(t) dt \quad (3-12)$$

$$t > t'$$

which, from (3-5) becomes

$$D_{gp}^*(t) = D_{gp}^*(t') - \frac{R_v^2}{R_{gp}^2} \left[\frac{C_1^*}{m_1} \left(e^{m_1(t-t')} - 1 \right) + \frac{c_2^*}{m_2} \left(e^{m_2(t-t')} - 1 \right) + c_3(t-t') \right] \quad (3-13)$$

$$t > t'$$

which is the analog of (2-17).

The pressure at the bottom of the melt pool away from the freeze valve is the analog of (2-18)

$$P_1^*(t) \cong \frac{\rho g}{144 g_0} D_g^*(t) \quad (3-14)$$

$$z_{ms} < z_{bgp} \quad t > t'$$

Computer Model

The theory explained in the previous section was encoded into a FORTRAN-IV program. This program uses data describing the physical properties of the fluid and the freeze valve dimensions as input and calculates instantaneous values of pool depth, fluid velocity through the valve, and mass flowrate through the valve. Appendix 1 contains a copy of the program and example of its output.

Verification of Theory

A small test station was built at the ETF to evaluate the validity of the isothermal freeze valve computer model. Silicone fluids were used instead of molten glass. Computer simulations of several tests were compared with data obtained during the experiments.

The test station consisted of an elevated holding tank 2-foot ID, 30-inches deep, with a flanged hole in the bottom (Figure 2). The flanged hole permitted the coupling of several different valve assemblies to the holding tank. A valve assembly consisted of a matching flange, a length of the appropriate diameter pipe, and a 1/4-turn ball valve. A catch tank was placed underneath the valve assembly to collect the draining fluids. Except for the PVC ball valves, all parts were constructed from 304L stainless steel.

Silicone fluid was chosen to simulate the molten glass. It is available in several viscosities at room temperature, including that of molten glass at 1150°C (10 to 150 poise). Silicone is also inert, noncorrosive, and nontoxic. Fluid with viscosities of 12 poise, 50 poise, and 75 poise were used during the experiment. Additional data on silicone fluids is provided in Appendix II.

The experimental procedure for each test consisted of three steps. First, a valve assembly was installed in the test station. The holding tank was then filled with silicone fluid. The fluid was allowed to sit for 30 minutes to allow air bubbles entrapped during tank filling to escape, and during this time period, the fluid temperature was measured (for determining the fluid density and viscosity). The ball valve was opened (to simulate the melting of the glass plug in an actual freeze valve). The time required to lower the fluid level in the holding tank one inch was recorded along with the tank depth. After the test was completed, a sample of the test fluid was analyzed by a HAAKE® viscometer to determine its viscosity.

Table I lists the independent variables held constant during each test. Hereafter, these tests will be referred to by case letter.

Figures 2 through 4 compare the results of the silicone fluid experiment with predictions generated by the computer model. The fluid level in the holding tank is plotted as a function of time. The good agreement seen in these figures demonstrate the validity of the model.

The theoretical effect of valve bottom pressure, P_{bv} , on freeze valve performance has not been experimentally verified. However, tests conducted on the small cylindrical melter 3 (SCM-2) jet controlled freeze valve demonstrate that it is possible to interrupt glass flow by pressurizing the valve bottom. An experimental program for testing this technique is being developed.

DISCUSSION

Figures 6 through 9 show the predicted effects of varying valve dimensions and glass viscosity on isothermal freeze valve performance. The "holding tank" dimensions coincide with those of the DWPF melt chamber (6-foot diameter, 34 inches of fluid). In each prediction, it is assumed that the pressure difference between the melter plenum and the bottom of the freeze valve is negligible. A 60 poise glass with a density of 2.4 g/cm, an arbitrary valve radius of 1 inch, and valve length of 1.5 foot were chosen as references for comparison.

The effect of valve radius on freeze valve performance is shown in Figures 5 and 6. From Figure 6 it is seen that increasing the valve radius results in a significant decrease in drain time. Theoretically, the drain time is inversely proportional to the square of the freeze valve radius. Figure 7 represents the mass flowrate of glass exiting the freeze valve as a function of valve radius and glass pool depth. The mass flowrate through the valve is directly proportional to the radius of the valve raised to the fourth power.

The length of the freeze valve influences both the driving force and the resistance force governing valve performance. The static head motivating glass flow increases with increasing valve length. However, viscous resistance to flow is also directly related to freeze valve length. As Figure 8 shows, the increase in viscous effect completely overwhelms the static head effect generated by lengthening the valve. Increasing the valve length for a fixed diameter valve results in lower mass flow rates.

Figure 9 shows the theoretical effect of glass viscosity on freeze valve performance. This figure demonstrates the practical limitations of the isothermal freeze valve model. The theory used in the model assumes that flow through the valve is laminar and that contraction losses are minimal. However, for large diameter valves and low viscosity fluids this assumption is not valid. The theoretical flowrate through the valve approaches infinity as the fluid viscosity approaches zero. To determine if the model can be applied to a specific situation, the predicted Reynolds number can be used to determine if the assumption of laminar flow is valid.

Figures 6 through 9 can be used to determine a suitable freeze valve size for the DWPF melter. Extrapolation from Figures 7 and 9 indicate that a 1-inch-ID freeze valve, 1.5 feet long, would drain the melter at an average rate of 1125 lb/hr (assuming a molten glass viscosity similar to Frit 168 + simulation-2*). A valve this size could drain a full melter near the end of its lifetime in about 13 hours (plus time required for canister changes).

* 80 poise at 1100°C

CONCLUSION

Uninterrupted, isothermal flow of molten glass through a freeze valve has been modeled. The accuracy of this model is limited to cases in which laminar flow is encountered. Tests with silicone fluids have verified the sufficient information for sizing the diameter of the DWPF freeze valve is available.

QUALITY ASSURANCE

All data acquisitions were performed in accordance with the Defense Waste Processing Section Quality Assurance Plan and DPST-QA-83-2-2, "Quality Assurance Review for the Small Cylindrical Melter-2." All experimental data were recorded in registered laboratory notebook DPSTN-4071.

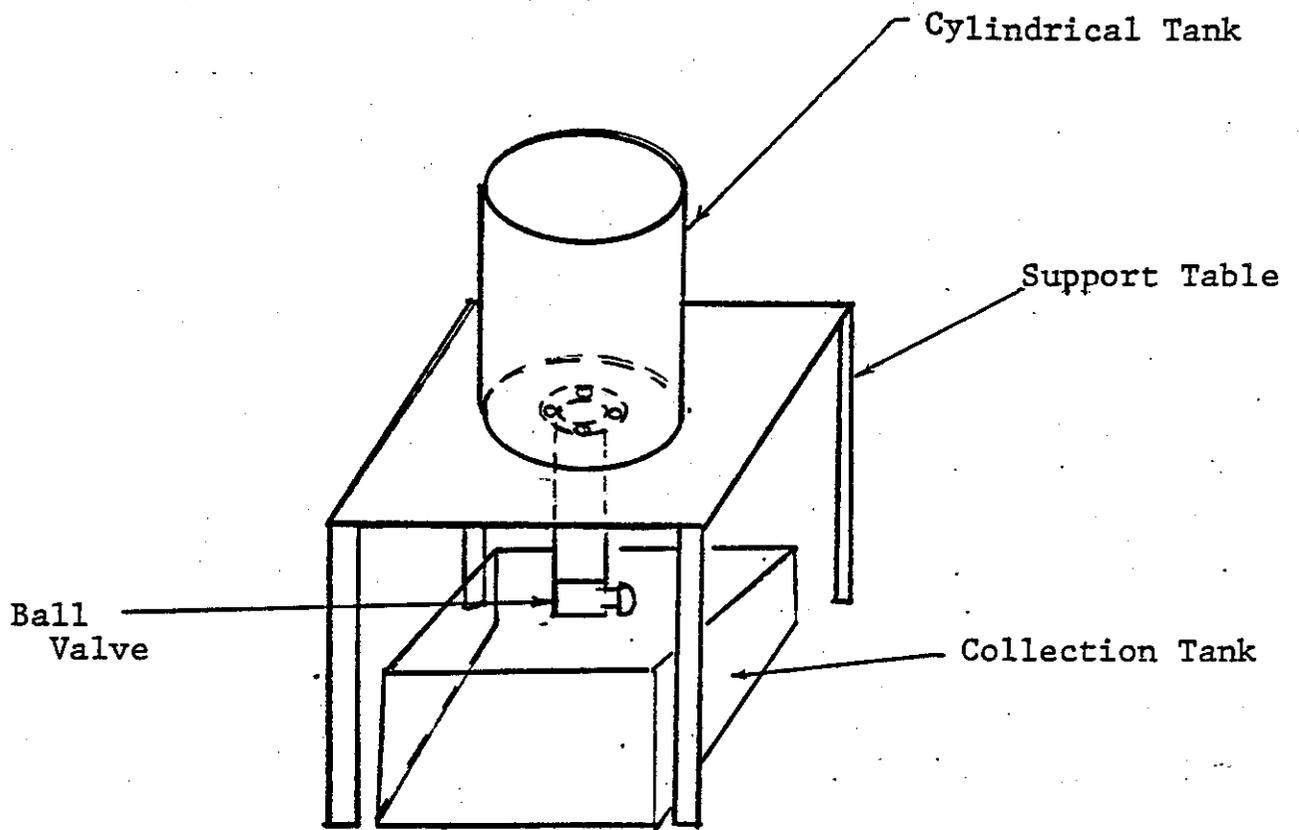


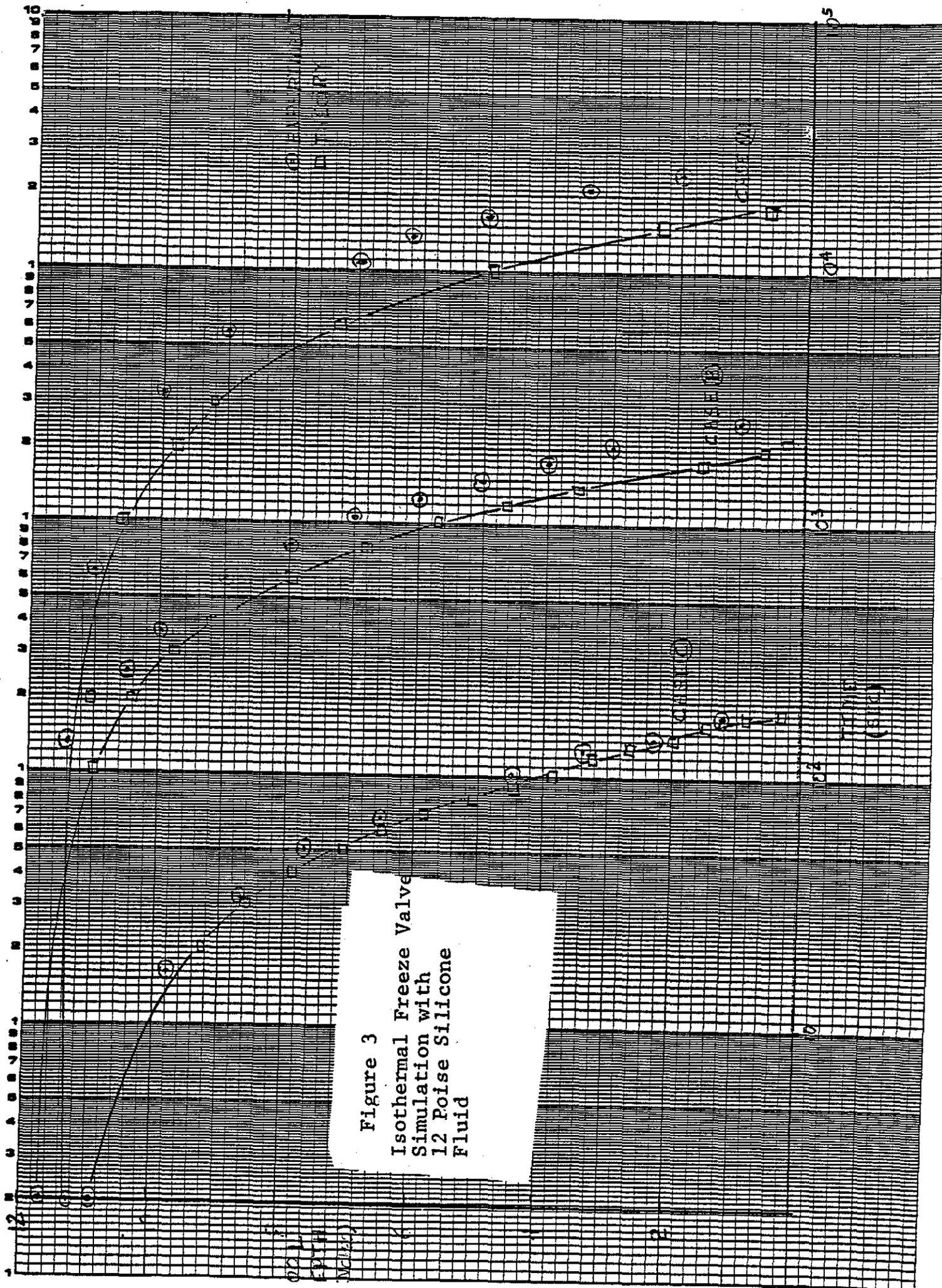
FIGURE 2

ISOTHERMAL FREEZE VALVE TEST STATION

TABLE 1

Dependent Variables for Experimental Tests

<u>Case</u>	<u>Valve ID, Inches</u>	<u>Valve Length, Inches</u>	<u>Fluid Viscosity, Poise</u>	<u>Initial Tank Depth, Inches</u>	<u>Fluid Density, g/cc</u>
A	0.639	24.5	48.4	11.25	0.994
B	1.113	22.5	47.8	11.75	0.994
C	2.125	24.5	51.4	10.875	0.906
D	0.639	24.5	12.66	9.0	1.102
E	1.113	22.5	12.66	8.75	1.102
F	2.125	24.5	12.66	9.0	0.967
G	1.113	22.5	76.74	10	0.969
H	2.125	24.5	76.74	11.25	76.74



0011
E11H
M1221

2

0

24
25
(15)

204

2

25

Figure 4
Isothermal Freeze Valve
Simulation With 50
Poise Silicone Fluid

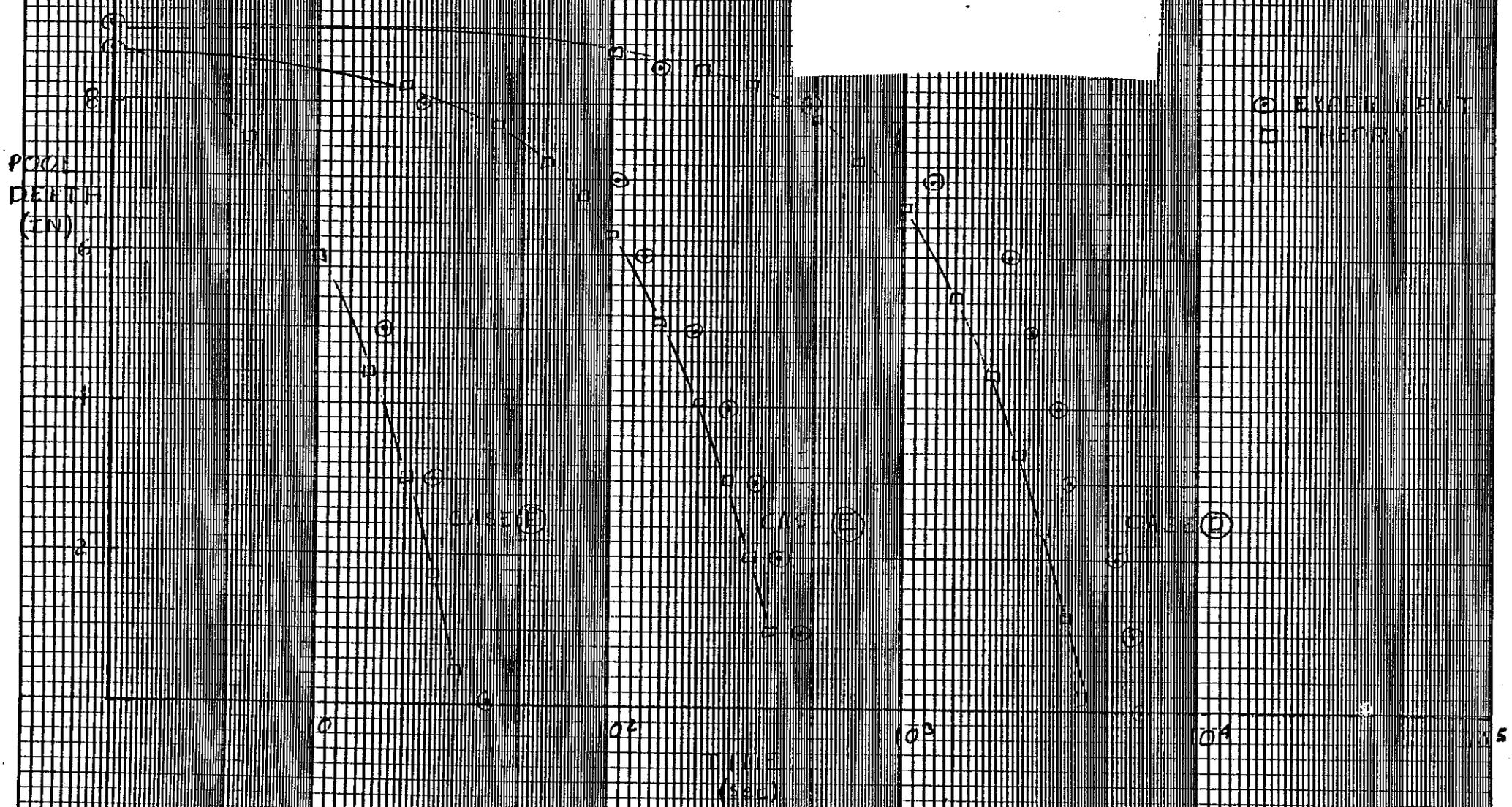


Figure 5
Isothermal Freeze Valve
Simulation With 75 Poise
Silicone Fluid

POOL
DEPTH
(In)

○ EXPERIMENTAL
□ THEORY

CASE (E)

CASE (F)

TIME
(SEC)

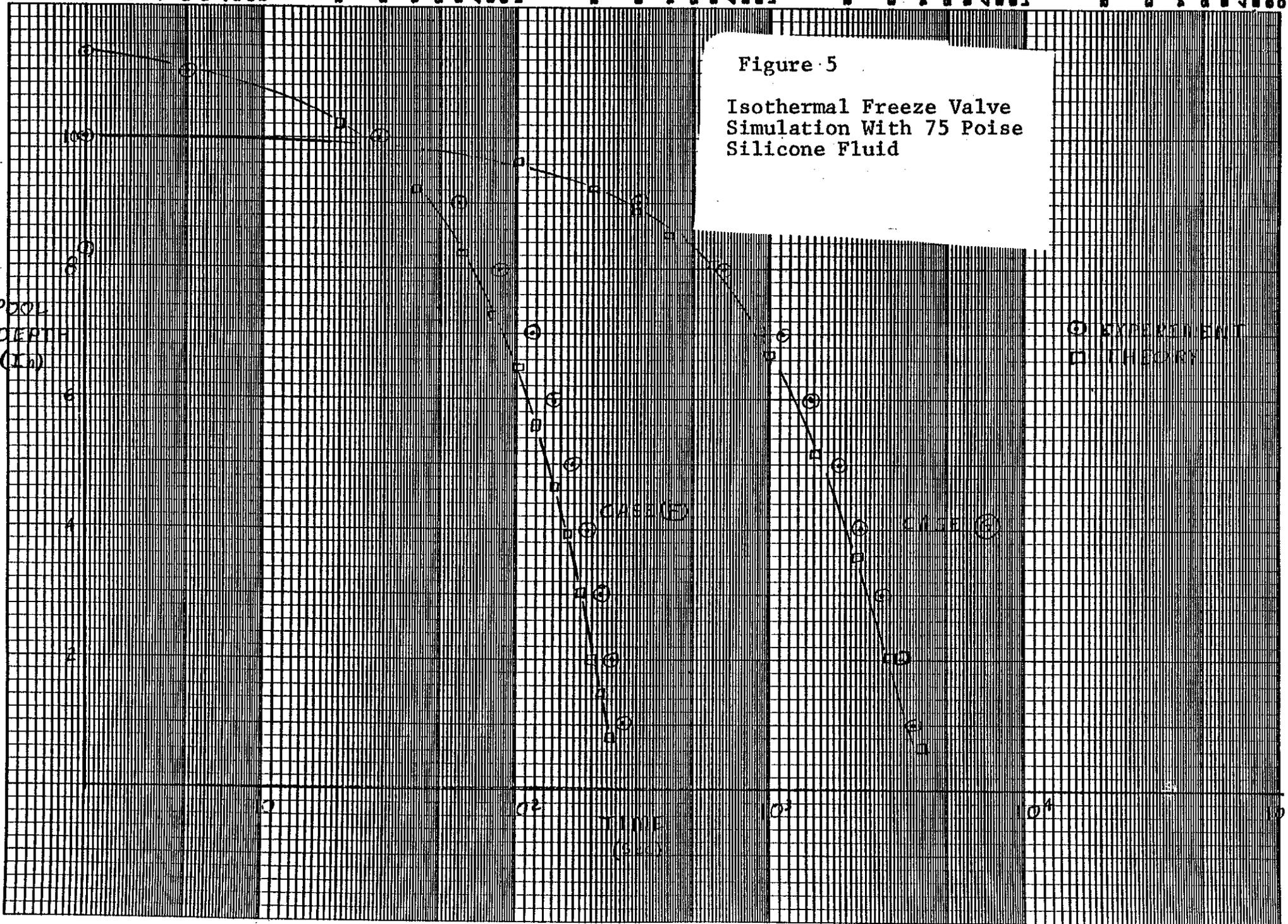


FIGURE 6

EFFECT OF VALVE RADIUS ON FREEZE VALVE PERFORMANCE

height vs. time

60 poise fluid, $R_{gp} = 3$ ft, $L_v = 1.5$ ft

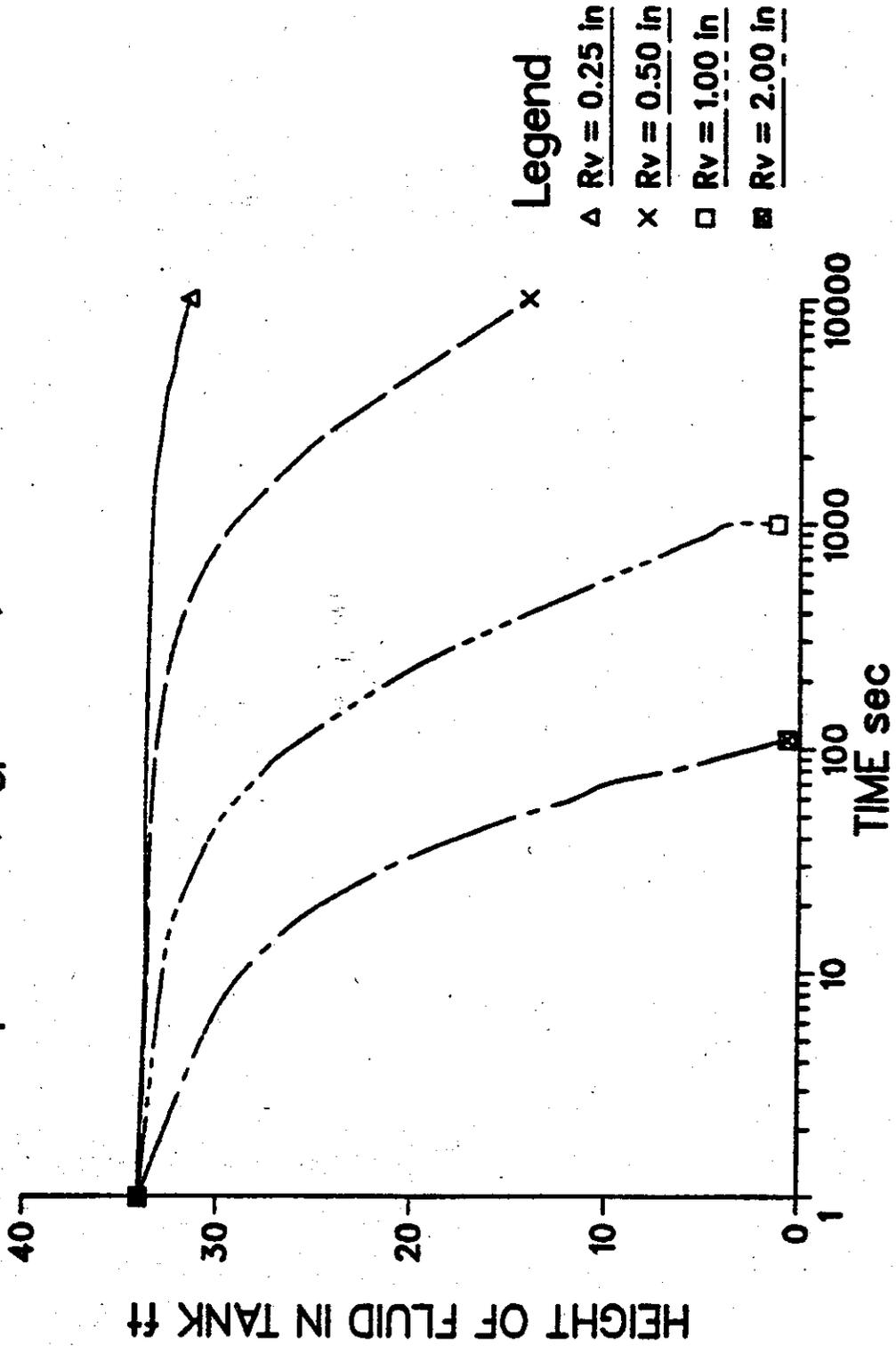


FIGURE 7

EFFECT OF VALVE RADIUS ON FREEZE VALVE PERFORMANCE

flow rate vs. glass pool depth
60 poise fluid, $R_{gp} = 3$ ft, $L_v = 1.5$ ft

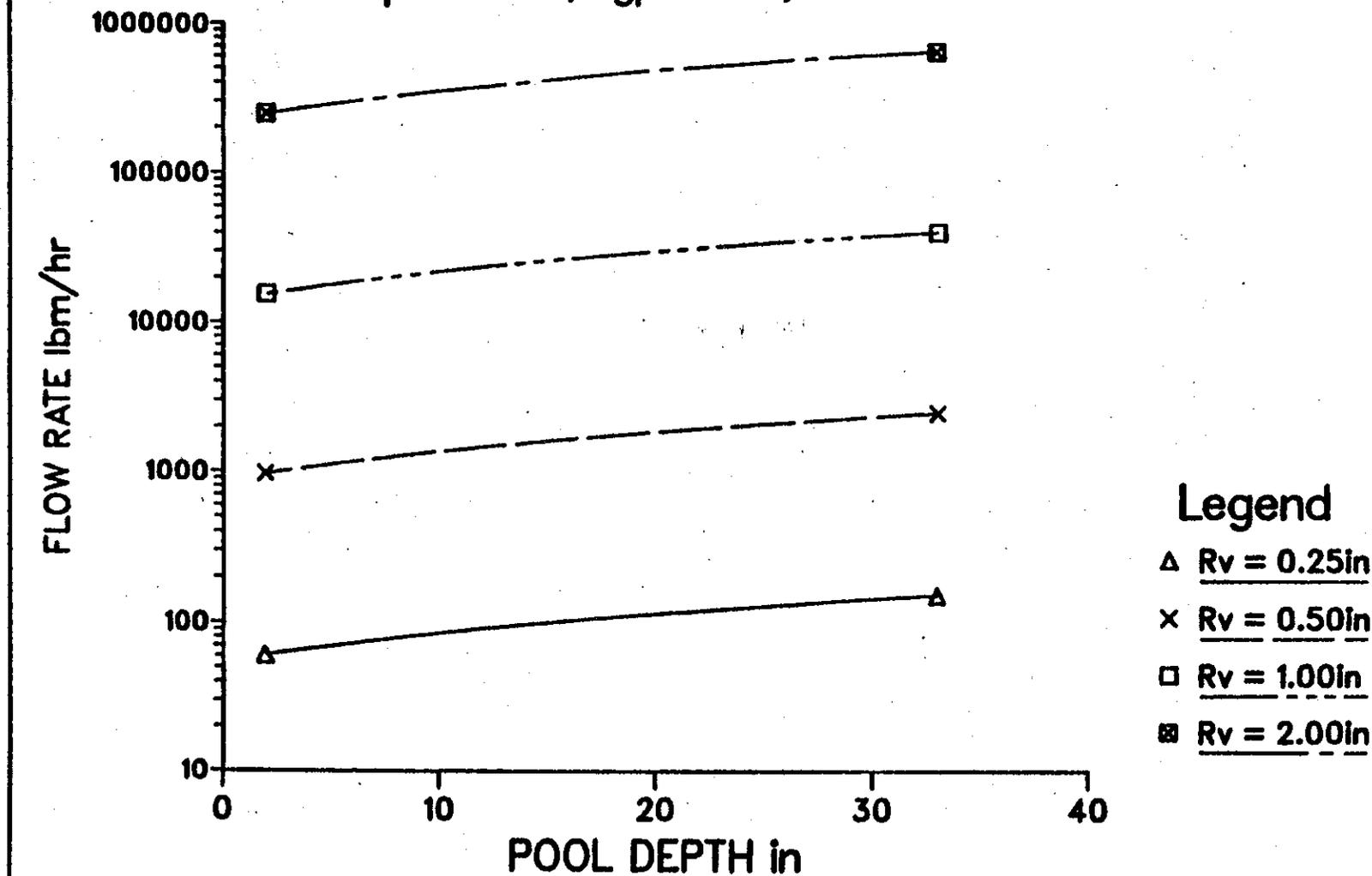


FIGURE 8

EFFECT OF GLASS VISCOSITY ON FREEZE VALVE PERFORMANCE
FLOW RATE VS. GLASS POOL DEPTH
 $R_{gp} = 3.0\text{ft}$, $R_v = 1.00\text{in}$, $L_v = 1.5\text{ft}$

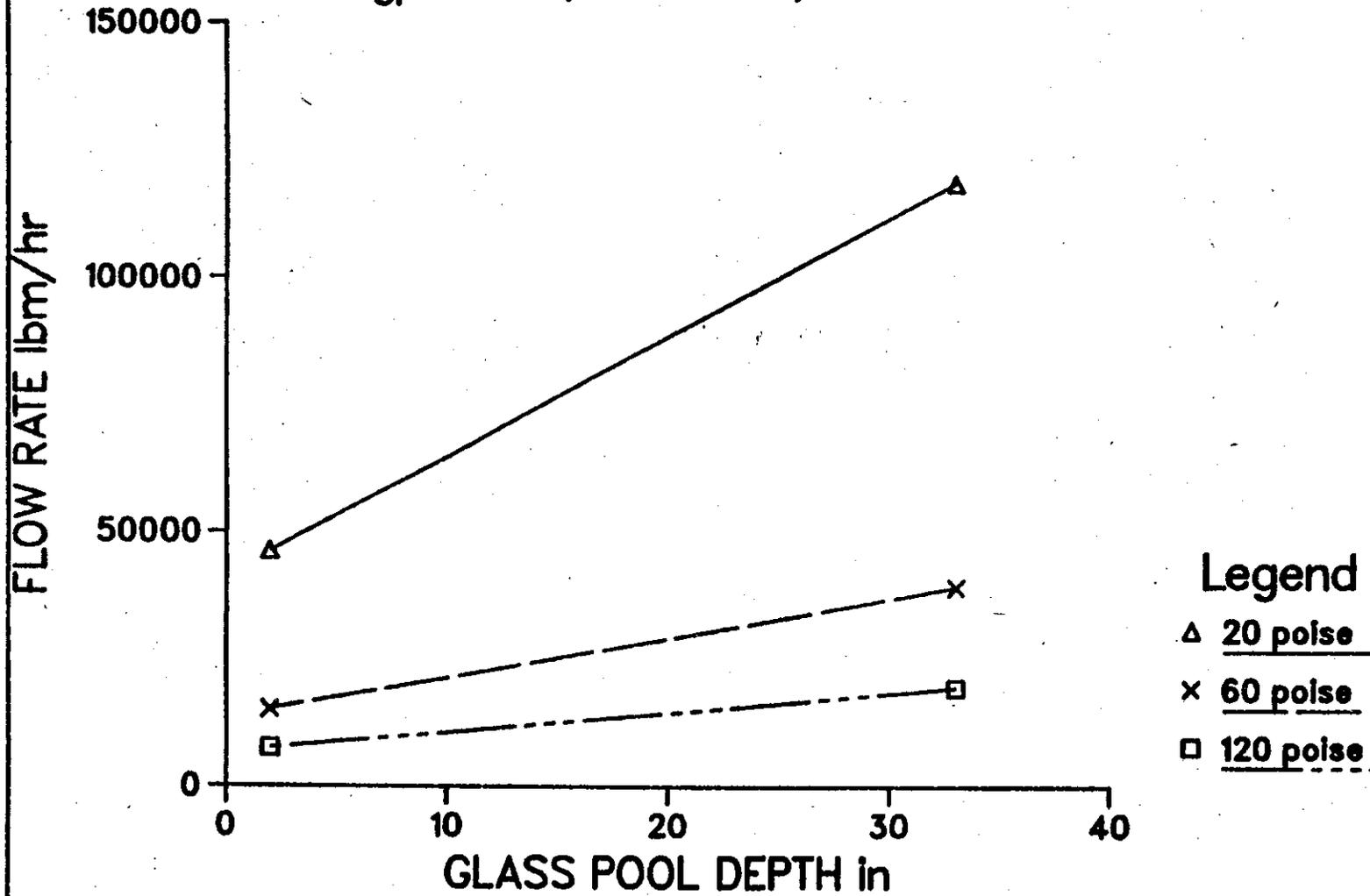
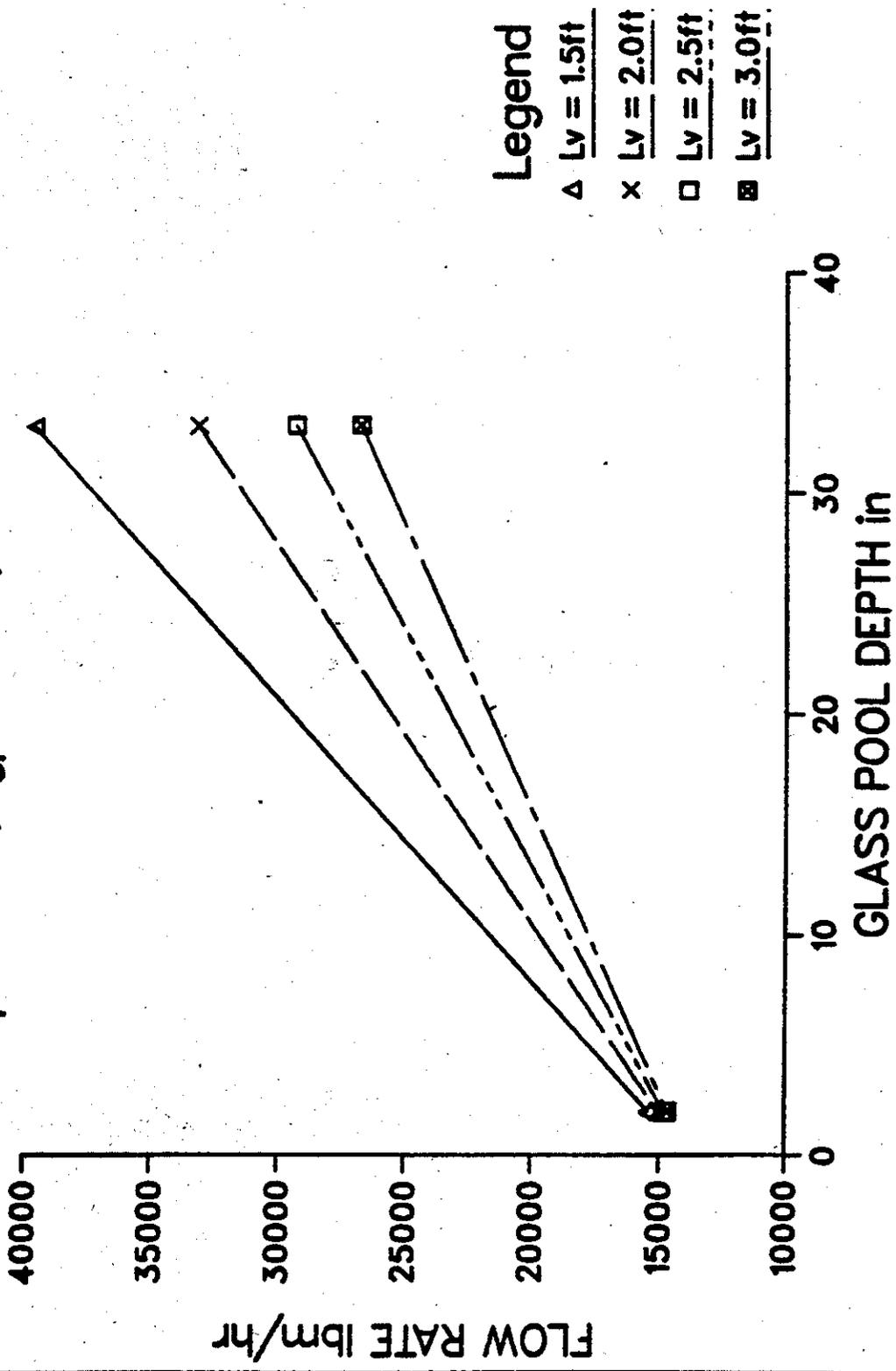


FIGURE 9

EFFECT OF VALVE LENGTH ON FREEZE VALVE PERFORMANCE

flow rate vs. glass pool depth
60 poise fluid, $R_{gp} = 3.0$ ft, $R_v = 1.00$ in



APPENDIX I

Fortran IV Computer Model of Isothermal Freeze Valve Theory

ISN 0005

C C C C C C

ISN 0006

INTEGER IFIN, START, KICK, J

L O A D D A T A

ISN 0007

DATA DELTAT/0.02 /, DPG/0.0/,
 *DPG1 /2.8 //,
 *LV/1.50 //,
 *PBV/14.7 //,
 *PP/14.70 //,
 *RGP/3.00 //,
 *RV/1.00 //,
 *RHO/152.0 //,
 *IMAX/35200.0 //,
 *VISC/12.15 //,
 *ZMS/0.00 //, ZMS1/0.0 //,
 *G0/32.2 //,
 *G/32.2 //,
 *ZBGP/2.8 //,
 *TIME1/0.00 //,
 *VLCTY1/0.00 //,
 *START/2 //, CIPAST/0.0 //, MIPAST/0.0 //, M2PAST/0.0 //,
 *STP/35200.0 //,
 *NTIME/0.0 //,
 *TIME/0.0 //, J/0 //, K/0 //

C C C C

INITIALIZE ALL NECESSARY VARIABLES

ISN 0008
 ISN 0009
 ISN 0010

RV = (RV/12.0)
 IFIN = (IMAX/DELTAT)
 KICK = DINT(10.0/DELTAT)
 P1 = PP + RHO*G*DPG1/(144.0*G0)
 P21 = P1

C

C PRINT HEADINGS AND INPUT VARIABLES

ISN 0013

WRITE (6,400) DELTAT, DPG1, LV, PBV, PP, RGP, RV, ZMS1,
 *IMAX, VISC, RHO

ISN 0014

ALPHA1 = RHO*(ZBGP-ZMS) - (3.0*RHO*VLCTY*
 DELTAT(RV/RGP)**2) + (2.0*RHO*VLCTY*DELTAT)
 * + RHO*LV

C C

ISN 0015

ALPHA2 = 8.0*VISC*LV/(RV**2)

ISN 0016

ALPHA3 = RHO*G*(RV/RGP)**2

ISN 0017

ALPHA4 = RHO*(VLCTY**2)*((RV/RGP)**2) -

01181060
 01190083
 01191083
 01200000
 01210000
 01220000
 01230000
 01240000
 01250000
 01260000
 01270000
 01280052
 01290089
 01300089
 01310000
 01320000
 01340089
 01341089
 01350089
 01360069
 01370089
 01380000
 01390000
 01400000
 01410089
 01420000
 01430000
 01431059
 01432069
 01433000
 01433104
 01433600
 01450000
 01460000
 01470000
 01480000
 01490000
 01491089
 01510000
 01520018
 01530000
 01540000
 01541000
 01542000
 01543000
 01544000
 01545000
 01545164
 01545264
 01545366
 01545482
 01545564
 01545689
 01545764
 01545864
 01545964
 01546064

	C		01546182
ISN 0018		TEST1 = (ALPHA2**2)-4.0*ALPHA1*ALPHA3	01546264
ISN 0019		IF(TEST1.LT.0.0) GO TO 40	01546364
	C		01546464
	C		01546564
	C	DETERMINE CONSTANTS	01546664
	C		01546764
ISN 0021		M1 = ((-ALPHA2) + (TEST1**0.5))/(2.0*ALPHA1)	01546864
	C		01546964
ISN 0022		M2 = ((-ALPHA2) - (TEST1**0.5))/(2.0*ALPHA1)	01547064
	C		01547164
	C		01547264
ISN 0023		C3 = ((PP-PBV)*144.0*G0 +ALPHA4)/ALPHA2	01547364
	C		01547464
ISN 0024		C1 = (1.0/(M1-M2))*(144.0*G0*(P21-PBV)/(RHO*LV) + *G + (C3*M2))	01547564
	C		01547664
ISN 0025		C2 = VLCTY1 -C1 -C3	01547764
ISN 0026		M1PAST = M1	01547864
ISN 0027		M2PAST = M2	01547964
ISN 0028		C1PAST = C1	01548064
ISN 0029		C2PAST = C2	01548164
	C		01548264
ISN 0030		WRITE (6,500)	01548364
	C	START MODEL CALCULATIONS	01548400
	C		01549000
ISN 0031		DO 1000 I = START,IFIN	01550000
	C	COMPUTATION OF ALPHA VALUES	01560000
	C		01570000
ISN 0032		TIME = TIME + DELTAT	01590000
ISN 0033		ANTIME = TIME - NTIME	01600000
	C		01610000
ISN 0034		ALPHA1= RHO*(ZBGP-ZMS) - (3.0*RHO*VLCTY* *DELTAT*((RV/RGP)**2)) + (2.0*RHO*VLCTY*DELTAT) * + RHO*LV	01620000
	C		01630018
	C		01640066
ISN 0035		ALPHA2 = 8.0*VISC*LV/(RV**2)	01641082
	C		01650000
ISN 0036		ALPHA3 = RHO*GX*((RV/RGP)**2)	01660089
	C		01670000
ISN 0037		ALPHA4 = RHO*(VLCTY**2)*(((RV/RGP)**2) - *1.0) + RHO*GX*(ZBGP-ZMS + LV)	01680000
	C		01690000
ISN 0038		TEST1 = (ALPHA2**2)-4.0*ALPHA1*ALPHA3	01700000
ISN 0039		IF(TEST1.LT.0.0) GO TO 40	01710082
	C		01750000
	C		01760000
	C		01780045
	C		01790000
	C	DETERMINE CONSTANTS	01800000
	C		01810000
ISN 0041		M1 = ((-ALPHA2) + (TEST1**0.5))/(2.0*ALPHA1)	01820000
	C		01830000
ISN 0042		M2 = ((-ALPHA2) - (TEST1**0.5))/(2.0*ALPHA1)	01840000
	C		01850000
	C		01860000
ISN 0043		C3 = ((PP-PBV)*144.0*G0 +ALPHA4)/ALPHA2	01970000
	C		01980000
	C		01990000

D E T E R M I N E V E L O C I T Y

D E T E R M I N E M E L T P O O L D E P T H

STOP COMPUTATION IF MELT CHAMBER IS EMPTY

CHECK FOR INPUT PRESSURIZATION

STEP INPUT BOUNDARY CONDITIONS

- 02000000
- 02010000
- 02020000
- 02030000
- 02040000
- 02050000
- 02130000
- 02140000
- 02150000
- 02151000
- 02152000
- 02153058
- 02154058
- 02171000
- 02171131
- 02171231
- 02172000
- 02180075
- 02190000
- 02200000
- 02201083
- 02202083
- 02210000
- 02220000
- 02230075
- 02240075
- 02250000
- 02261003
- 02270000
- 02280058
- 02310000
- 02311058
- 02312058
- 02313058
- 02314058
- 02320000
- 02330045
- 02340000
- 02350000
- 02350100
- 02350200
- 02350339
- 02350400
- 02350500
- 02350600
- 02351004
- 02352000
- 02360000
- 02370000
- 02371000
- 02372000
- 02373040
- 02374000
- 02380000
- 02381000
- 02383000
- 02384000

C
C
C
C
ISN 0096 STOP
ISN 0097 END

03430000
03440000
03460000
03470000
03480000
03490000
03500000

/ MAIN / SIZE OF PROGRAM 000C24 HEXADECIMAL BYTES

NAME	TAG	TYPE	ADD.												
G	F	R*8	0004C8	I	SF	I*4	0004A0	J		I*4	NR	K	SF	I*4	0004A4
Q	SF	R*8	0004D0	C1	SF	R*8	0004D8	C2	SF	R*8	0004E0	C3	SF	R*8	0004E8
GO	F	R*4	0004A8	GO	F	R*8	0004F0	LV	F	R*8	0004F8	M1	SF	R*8	000500
M2	SF	R*8	000508	PP	SF	R*8	000510	PV		R*8	NR	P1	SF	R*8	000518
P2	SF	R*8	000520	RV	SF	R*4	0004AC	APG	SF	R*8	000528	DPG	SF	R*8	000530
PBV	SF	R*4	0004B0	PRV		R*8	NR	P21	SF	R*8	000538	RGP	F	R*8	000540
RHO	F	R*8	000548	SM1	SFA	R*8	000550	SM2	SFA	R*8	000558	SP2		R*8	NR
STP		R*8	000560	ZMS	SF	R*8	000568	DEXP	F	XF	000000	DPG1	SF	R*8	000570
IFIN	SF	I*4	0004B4	KICK	S	I*4	0004B8	SDPG		R*8	NR	TIME	SF	R*8	000578
TMAX	F	R*8	000580	VISC	F	R*8	000588	ZBGP	F	R*8	000590	ZMS1	F	R*8	000598
NTIME	SF	R*8	0005A0	START	F	I*4	0004BC	STIME		R*8	NR	TEST1	SF	R*4	0004C0
TIME1		R*8	NR	VLCTY	SF	R*8	0005A8	FRXPR#		XF	000000	ALPHA1	SF	R*8	0005B0
ALPHA2	SF	R*8	0005B8	ALPHA3	SF	R*8	0005C0	ALPHA4	SF	R*8	0005C8	ANTIME	SF	R*8	0005D0
C1PAST	SF	R*8	0005D8	C2PAST	SF	R*8	0005E0	DELTAT	FA	R*8	0005E8	IBCOM#	F	XF	000000
M1PAST	SF	R*8	0005F0	M2PAST	SF	R*8	0005F8	SVLCTY		R*8	NR	VLCTY1	SF	R*8	000600

SOURCE STATEMENT LABELS

LABEL	ISN	ADDR	LABEL	ISN	ADDR	LABEL	ISN	ADDR	LABEL	ISN	ADDR
998	61	000AFA	999	74	000B4E	1000	80	000B9C	35	81	000BA6
40	82	000BA8	4000	88	000BF8 NR						

COMPILER GENERATED LABELS

LABEL	ISN	ADDR									
100000	1	0006A0	100001	21	00084A	100002	32	0008FC	100003	41	000982
100004	49	000A28	100005	50	000A30	100006	51	000A3C	100007	52	000A44
100008	64	000B10	100009	66	000B1A	100010	68	000B26	100011	77	000B58

FORMAT STATEMENT LABELS

LABEL	ISN	ADDR	LABEL	ISN	ADDR	LABEL	ISN	ADDR	LABEL	ISN	ADDR
300	89	000028 NR	400	90	000050	500	91	000245	600	92	00029E
601	93	0002AC NR	910	94	0002B7 NR	950	95	0002CD NR			

NUMBER LEVEL FORTRAN H EXTENDED ERROR MESSAGES

E307I 4(W) NAME J THE DATA STATEMENT CONTAINS A VARIABLE THAT IS NOT REFERENCED.
E307I 4(W) NAME TIME1 THE DATA STATEMENT CONTAINS A VARIABLE THAT IS NOT REFERENCED.
E610I 4(W) LABEL 4000 THE STATEMENT NUMBER OR GENERATED LABEL IS UNREACHABLE.
PTIONS IN EFFECT*NAME(MAIN) OPTIMIZE(3) LINECOUNT(58) SIZE(MAX) AUTODBL(NONE)
PTIONS IN EFFECT*SOURCE EBCDIC NOLIST NODECK OBJECT MAP NOFORMAT NOGOSTMT NOXREF NOALC NOANSF NOTERM IBM FLAG(I) XL
STATISTICS* SOURCE STATEMENTS = 96, PROGRAM SIZE = 3108, SUBPROGRAM NAME = MAIN
STATISTICS* 3 DIAGNOSTICS GENERATED, HIGHEST SEVERITY CODE IS 4

**** END OF COMPILATION ****

672K BYTES OF CORE NOT USED

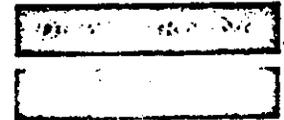
DELTA T = 2.000D-02 SECONDS
INITIAL DEPTH OF MELT POOL = 2.800D+00 FEET
VALVE LENGTH = 1.500D+00 FEET
PRESSURE AT BOTTOM OF FREEZE VALVE = 1.470E+01 PSIA
ABSOLUTE PLENUM PRESSURE = 1.470D+01 PSIA
RADIUS OF GLASS POOL = 3.000D+00 FEET
RADIUS OF FREEZE VALVE INNER PIPE = 8.333E-02 FEET
REFERENCE DISTANCE TO MELT SURFACE = 0.0 FEET
DEFAULT MAXIMUM EXECUTION TIME = 3.520D+04 SECONDS
GLASS VISCOSITY = 1.215D+01 LBM/FT-SEC
GLASS DENSITY = 1.520D+02 LBM/CU.FT.

P-R-I-M-A-R-Y R-E-S-P-O-N-S-E V-A-R-I-A-B-L-E-S

TIME	VLCTY	DPG	Q
1.000D+01	2.932D+00	3.333D+01	3.500D+04
2.000D+01	2.912D+00	3.306D+01	3.476D+04
3.000D+01	2.891D+00	3.280D+01	3.452D+04
4.000D+01	2.871D+00	3.253D+01	3.428D+04
5.000D+01	2.852D+00	3.228D+01	3.404D+04
6.000D+01	2.832D+00	3.202D+01	3.381D+04
7.000D+01	2.813D+00	3.177D+01	3.358D+04
8.000D+01	2.793D+00	3.152D+01	3.335D+04
9.000D+01	2.774D+00	3.127D+01	3.312D+04
1.000D+02	2.755D+00	3.103D+01	3.289D+04
1.100D+02	2.737D+00	3.079D+01	3.267D+04
1.200D+02	2.718D+00	3.055D+01	3.245D+04
1.300D+02	2.700D+00	3.031D+01	3.223D+04
1.400D+02	2.682D+00	3.008D+01	3.202D+04
1.500D+02	2.664D+00	2.985D+01	3.180D+04
1.600D+02	2.646D+00	2.962D+01	3.159D+04
1.700D+02	2.628D+00	2.940D+01	3.138D+04
1.800D+02	2.611D+00	2.918D+01	3.117D+04
1.900D+02	2.594D+00	2.896D+01	3.096D+04
2.000D+02	2.576D+00	2.874D+01	3.076D+04
2.100D+02	2.559D+00	2.853D+01	3.055D+04
2.200D+02	2.543D+00	2.832D+01	3.035D+04
2.300D+02	2.526D+00	2.811D+01	3.015D+04
2.400D+02	2.509D+00	2.790D+01	2.996D+04
2.500D+02	2.493D+00	2.769D+01	2.976D+04
2.600D+02	2.477D+00	2.749D+01	2.957D+04
2.700D+02	2.461D+00	2.729D+01	2.938D+04
2.800D+02	2.445D+00	2.709D+01	2.919D+04
2.900D+02	2.429D+00	2.690D+01	2.900D+04
3.000D+02	2.413D+00	2.670D+01	2.881D+04
3.100D+02	2.398D+00	2.651D+01	2.863D+04
3.200D+02	2.382D+00	2.632D+01	2.844D+04
3.300D+02	2.367D+00	2.614D+01	2.826D+04
3.400D+02	2.352D+00	2.595D+01	2.808D+04
3.500D+02	2.337D+00	2.577D+01	2.790D+04
3.600D+02	2.322D+00	2.559D+01	2.772D+04
3.700D+02	2.308D+00	2.541D+01	2.755D+04
3.800D+02	2.293D+00	2.523D+01	2.737D+04
3.900D+02	2.279D+00	2.505D+01	2.720D+04
4.000D+02	2.264D+00	2.488D+01	2.703D+04
4.100D+02	2.250D+00	2.471D+01	2.686D+04
4.200D+02	2.236D+00	2.454D+01	2.669D+04
4.300D+02	2.222D+00	2.437D+01	2.653D+04
4.400D+02	2.208D+00	2.420D+01	2.636D+04
4.500D+02	2.195D+00	2.404D+01	2.620D+04
4.600D+02	2.181D+00	2.388D+01	2.604D+04
4.700D+02	2.167D+00	2.371D+01	2.588D+04
4.800D+02	2.154D+00	2.355D+01	2.572D+04
4.900D+02	2.141D+00	2.340D+01	2.556D+04
5.000D+02	2.128D+00	2.324D+01	2.540D+04
5.100D+02	2.115D+00	2.309D+01	2.525D+04
5.200D+02	2.102D+00	2.293D+01	2.509D+04
5.300D+02	2.089D+00	2.278D+01	2.494D+04
5.400D+02	2.076D+00	2.263D+01	2.479D+04
5.500D+02	2.064D+00	2.248D+01	2.464D+04
5.600D+02	2.051D+00	2.234D+01	2.449D+04
5.700D+02	2.145D+00	2.193D+01	2.560D+04

5.800D+02	2.134D+00	2.178D+01	2.547D+04
5.900D+02	2.123D+00	2.163D+01	2.534D+04
6.000D+02	2.112D+00	2.148D+01	2.521D+04
6.100D+02	2.101D+00	2.133D+01	2.508D+04
6.200D+02	2.091D+00	2.118D+01	2.496D+04
6.300D+02	2.080D+00	2.104D+01	2.483D+04
6.400D+02	2.070D+00	2.089D+01	2.471D+04
6.500D+02	2.059D+00	2.075D+01	2.458D+04
6.600D+02	2.049D+00	2.061D+01	2.446D+04
6.700D+02	2.039D+00	2.047D+01	2.434D+04
6.800D+02	2.028D+00	2.033D+01	2.422D+04
6.900D+02	2.018D+00	2.020D+01	2.409D+04
7.000D+02	2.008D+00	2.006D+01	2.398D+04
7.100D+02	1.998D+00	1.993D+01	2.386D+04
7.200D+02	1.989D+00	1.979D+01	2.374D+04
7.300D+02	1.979D+00	1.966D+01	2.362D+04
7.400D+02	1.969D+00	1.953D+01	2.351D+04
7.500D+02	1.959D+00	1.940D+01	2.339D+04
7.600D+02	1.950D+00	1.927D+01	2.328D+04
7.700D+02	1.940D+00	1.914D+01	2.316D+04
7.800D+02	1.931D+00	1.901D+01	2.305D+04
7.900D+02	1.922D+00	1.889D+01	2.294D+04
8.000D+02	1.912D+00	1.876D+01	2.283D+04
8.100D+02	1.903D+00	1.864D+01	2.272D+04
8.200D+02	1.894D+00	1.852D+01	2.261D+04
8.300D+02	1.885D+00	1.839D+01	2.250D+04
8.400D+02	1.876D+00	1.827D+01	2.239D+04
8.500D+02	1.867D+00	1.815D+01	2.229D+04
8.600D+02	1.858D+00	1.804D+01	2.218D+04
8.700D+02	1.849D+00	1.792D+01	2.207D+04
8.800D+02	1.840D+00	1.780D+01	2.197D+04
8.900D+02	1.832D+00	1.769D+01	2.186D+04
9.000D+02	1.823D+00	1.757D+01	2.176D+04
9.100D+02	1.814D+00	1.746D+01	2.166D+04
9.200D+02	1.806D+00	1.735D+01	2.156D+04
9.300D+02	1.797D+00	1.723D+01	2.146D+04
9.400D+02	1.789D+00	1.712D+01	2.136D+04
9.500D+02	1.780D+00	1.701D+01	2.126D+04
9.600D+02	1.772D+00	1.690D+01	2.116D+04
9.700D+02	1.764D+00	1.680D+01	2.106D+04
9.800D+02	1.756D+00	1.669D+01	2.096D+04
9.900D+02	1.748D+00	1.658D+01	2.086D+04
1.000D+03	1.740D+00	1.648D+01	2.077D+04
1.010D+03	1.732D+00	1.637D+01	2.067D+04
1.020D+03	1.724D+00	1.627D+01	2.058D+04
1.030D+03	1.716D+00	1.616D+01	2.048D+04
1.040D+03	1.708D+00	1.606D+01	2.039D+04
1.050D+03	1.700D+00	1.596D+01	2.030D+04
1.060D+03	1.692D+00	1.586D+01	2.020D+04
1.070D+03	1.685D+00	1.576D+01	2.011D+04
1.080D+03	1.677D+00	1.566D+01	2.002D+04
1.090D+03	1.669D+00	1.556D+01	1.993D+04
1.100D+03	1.662D+00	1.547D+01	1.984D+04
1.110D+03	1.654D+00	1.537D+01	1.975D+04
1.120D+03	1.647D+00	1.527D+01	1.966D+04
1.130D+03	1.640D+00	1.518D+01	1.957D+04
1.140D+03	1.632D+00	1.508D+01	1.948D+04
1.150D+03	1.625D+00	1.499D+01	1.940D+04
1.160D+03	1.618D+00	1.490D+01	1.931D+04
1.170D+03	1.610D+00	1.480D+01	1.923D+04

Information about Silicone Fluids



STEVE R. ROLKA
1-800-243-2345

DESCRIPTION

Dow Corning 200 fluid is a water-clear silicone fluid available in viscosities ranging from 0.65 centistokes to 100,000 centistokes. Important features of this fluid include:

- Little change in physical properties over a wide temperature span — a relatively flat viscosity-temperature slope, and serviceability from -40 to over 400 F (-40 to 204 C)
- Excellent water repellency
- Good dielectric properties over a wide range of temperatures and frequencies
- Low surface tension — readily wets clean surfaces to impart water repellency and release characteristics
- Low-toxicity — tests have established that Dow Corning 200 fluid is essentially non-toxic and non-irritating (although temporary discomfort may result if rubbed into the eye)

Dow Corning 200 fluid also exhibits heat stability, oxidation resistance, very low vapor pressures, and high flash points. It is insoluble in organic liquids other than active solvents, and is nongreasy, nonrancidifying, and virtually odorless.

USES

Its unique combination of outstanding properties suit Dow Corning 200 fluid for a variety of application functions, as well as for a wide range of products and processes.

As a Release Material

Used along or as part of a compounded formula, Dow Corning 200 fluid provides an odorless,

DOW CORNING® 200 FLUID

(Approved for Federal Specification VV-D-1078A)

Type	Dimethyl siloxane polymer
Physical Form	Fluids with viscosities ranging from 0.65 to 100,000 centistokes
Special Properties	Thermal stability; water repellency; high dielectric strength; low surface tension; essentially nontoxic
Primary Uses	As a mechanical or electrical fluid, lubricant, antifoam, surfactant, chemical specialty additive

nontoxic, non-carbonizing mold release for rubber, plastics, and metal die castings.

As a Foam Preventive

Extremely small amounts of the fluid effectively control foam in many processing operations, especially in nonaqueous systems.

As a Mechanical Fluid

Excellent viscosity-temperature characteristics, thermal and chemical stability, shear-breakdown resistance, compressibility, and rubber compatibility make Dow Corning 200 fluid outstanding for mechanical/hydraulic uses. Typical uses include hydraulic stabilizers, damping mediums.

As a Surface-Active Material

Added to vinyl plastisols and liquid springs, Dow Corning 200 fluid improves the flow characteristics, de-aerates and lubricates the surface of the completed part.

As a Lubricant

The fluid provides excellent lubrication for plastic and elastomeric surfaces.

In Cosmetics and Skin Preparations

Dow Corning 200 fluid is an important ingredient in hand creams, skin protectants, suntan lotions, and hair grooming aids because it forms

a non-greasy, protective film which resists water and waterborne irritants, yet allows the skin to breathe. Literature is available that details the use of the fluid in cosmetics.

In Polishes and Chemical Specialities

Dow Corning 200 fluid is used in most automobile and furniture polishes for its ease of application, high gloss with minimum rubbing, and a durable water-repellent film. It is also used in many other specialty formulations, including aerosol starches and fabric conditioners. Additional literature is available.

In Electrical/Electronic Equipment

With excellent dielectric properties, Dow Corning 200 fluid is widely used for both insulating and damping applications. Refer to specific literature on Dow Corning 200 fluid, electrical-grade and electronic-grade.

In Food Processing

Noncontaminating and nonadulterating, Dow Corning 200 fluid is suited to many food-processing applications. When considering the fluid for uses involving FDA regulations, refer to literature on Food-Grade Dow Corning 200 fluid.

SOLUBILITY

The solubility of Dow Corning 200 fluid varies somewhat according to the viscosity chosen. The low viscosity grades are more completely soluble in a given solvent than are the higher viscosity grades. Since solubility varies, testing is recommended before attempting volume operations. Solubility of the silicone fluid in a number of commonly-used solvents is indicated below. Flammability and toxicity should also be important considerations in the choice of a solvent.

Solvents

Amyl acetate
Benzene
Carbon tetrachloride
Chloroethene NU*
Cyclohexane
Diesel fuel
Ethylene dichloride
Ethyl ether
2-Ethyl hexanol
Gasoline
Hexyl ether
Iso-octane
JP-4 jet fuel
Kerosene
Methyl ethyl ketone

Methylene chloride
Methyl ether
Mineral seal oil
Naphtha VM&P
Perchloroethylene
Stoddard solvent
Toluene
Trichloroethylene
Turpentine
Xylene

Partial Solvents †

Acetone
Butanol
Dioxane
Ethanol

Heptadecanol
Isopropanol

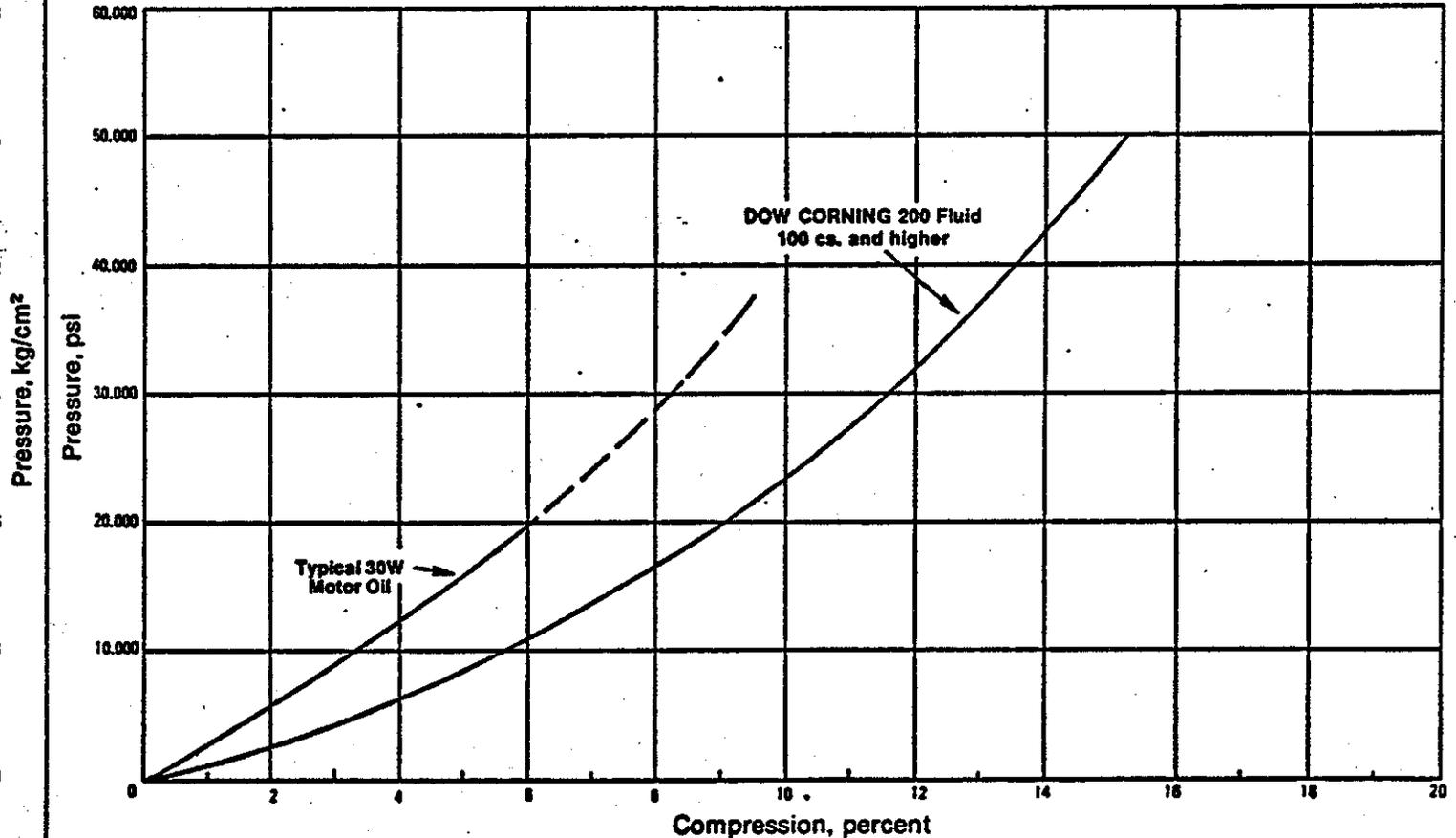
Nonsolvents

Cyclohexanol
Dimethylphthalate
Dodecanol
Dowanol* DE
Dowanol* EE
Ethylene glycol
Methanol
Paraffin oil
Propylene glycol
Water

* The Dow Chemical Company

† Partial Solvents— for lower viscosity grades

Compressibility* of Dow Corning 200 Fluid



* Corporate test method CTM 0490.

TYPICAL PROPERTIES OF STANDARD VISCOSITY GRADES¹

These values are not intended for use in preparing specifications.

Viscosity ² at 77 F (25 C), centistokes	Flash Point, closed cup ³	Pour Point ⁵	Specific Gravity ⁶ at 77 F (25 C), spindle reading	Viscosity Temperature Coefficient ⁷	Coefficient of Expansion, ⁸ cc/cc/C	Refractive Index ⁹ at 77 F (25
0.65	0 F (-18 C)	-90 F (-68 C)	0.761	0.31	0.00134	1.375
1.0	92 F (33 C)	-148 F (-100 C)	0.818	0.41	0.00134	1.382
1.5	130 F (55 C)	-148 F (-100 C)	0.853	0.46	0.00134	1.388
2.0	189 F (87 C)	-148 F (-100 C)	0.873	0.48	0.00117	1.390
5.0	275 F (135 C)	-148 F (-100 C)	0.920	0.55	0.00105	1.397
10	325 F (163 C)	-148 F (-100 C)	0.934	0.58	0.00108	1.399
20	450 F (232 C)	-121 F (-84 C)	0.949	0.59	0.00107	1.400
50	545 F (285 C)	-94 F (-70 C)	0.960	0.59	0.00104	1.401
100	600 F (315 C)	-85 F (-65 C)	0.960	0.60	0.00096	1.402
200	600 F (315 C)	-85 F (-65 C)	0.970	0.60	0.00096	1.403
350	600 F (315 C)	-85 F (-65 C)	0.970	0.60	0.00096	1.403
500	600 F (315 C)	-58 F (-50 C)	0.971	0.60	0.00096	1.403
1000	610 F (321 C)	-58 F (-50 C)	0.971	0.61	0.00096	1.403
12500	610 F (321 C)	-51 F (-46 C)	0.975	0.61	0.00096	1.403
30000	610 F (321 C)	-47 F (-43 C)	0.975	0.61	0.00096	1.403
60000	610 F (321 C)	-42 F (-41 C)	0.976	0.61	0.00096	1.403
100000	610 F (321 C)	-28 F (-33 C)	0.977	0.61	0.00096	1.403

¹ Although only standard viscosity grades are listed, other viscosities may be made available on request.

² CTM 0004

³ CTM 0021

⁴ CTM 0006

⁵ CTM 0133. Due to the effects of supercooling, this test method yields pour points lower than the temperatures at which these silicone fluids solidify when held at such temperature for a longer period.

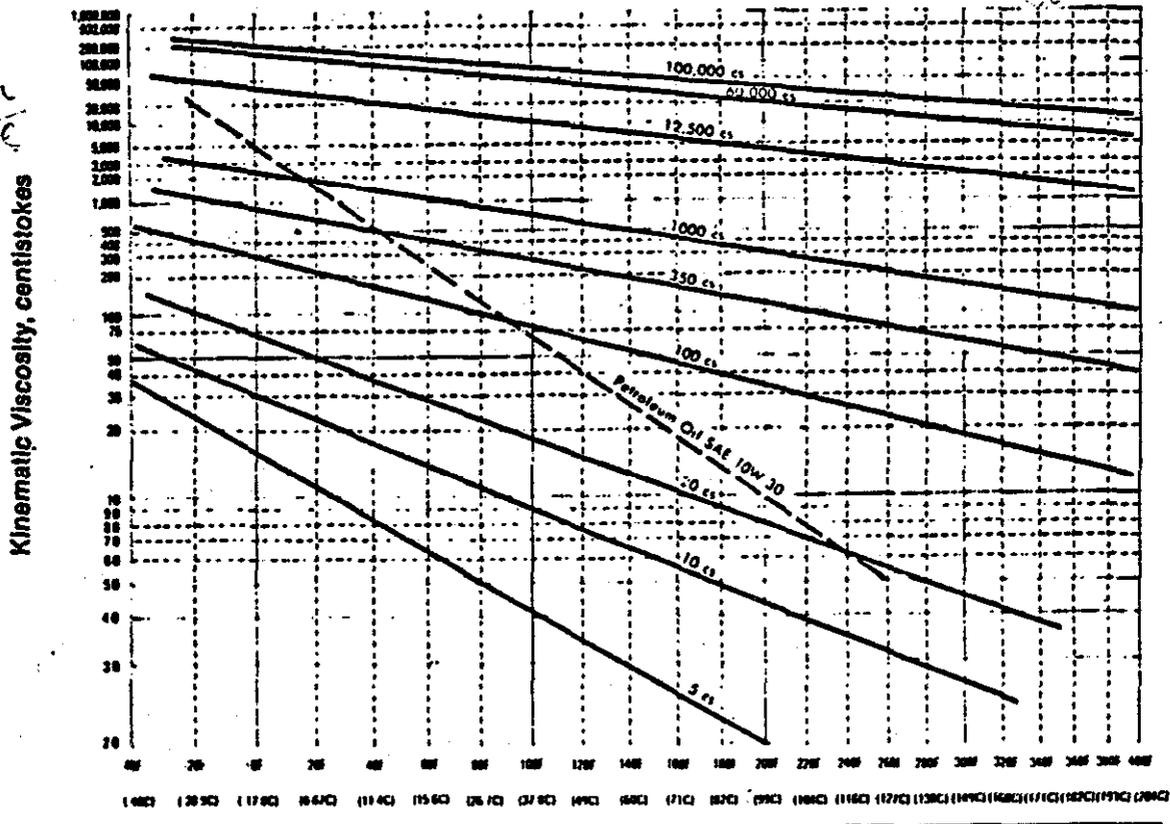
⁶ CTM 001A

$$\rho_{77}^{20} = 60.224 \text{ g/cc}^3$$

$$= 0.99589 \text{ g/cc}$$

10F \Rightarrow 1223 cSt \Rightarrow 10
 50F \Rightarrow 5157 cSt \Rightarrow 50
 100F \Rightarrow 10577 cSt \Rightarrow 100
 150F \Rightarrow 15733 cSt \Rightarrow 150

Viscosity Temperature Chart* for Dow Corning 200 Fluid



$$\eta = \frac{\mu}{\rho}$$

Surface viscosity F in/cm	Thermal Conductivity ¹¹ at 122 F (50 C)	Boiling Point	Specific Heat, cal/gm/C		
			25 C	100 C	200 C
5.9	0.00024	211 F (100 C) at 760 mm	----	----	----
7.4	----	305 F (152 C) at 760 mm	----	----	----
8.0	----	377 F (192 C) at 760 mm	0.410	0.437	0.474
8.7	0.00026	158-212 F (70-100 C) at 0.5 mm	----	----	----
9.7	----	248-320 F (120-160 C) at 0.5 mm	----	----	----
VOLATILITY ¹²					
% max weight loss after 24 hrs at 302 F (150 C)					
0.1	0.00032	10	----	----	----
0.6	0.00034	10	----	----	----
0.8	----	2.0	----	----	----
0.9	0.00037	0.5	----	----	----
1.0	----	0.5	----	----	----
1.1	0.0038	0.5	0.379	0.405	0.440
1.1	----	0.5	----	----	----
1.2	0.00038	0.5	----	----	----
1.5	0.00037	2.0	----	----	----
1.5	----	2.0	----	----	----
1.5	----	2.0	----	----	----
1.5	----	2.0	----	----	----

CTM 0747,1 — Viscosity at 210 F (99 C)
 Viscosity at 100 F (38 C)

¹² CTM 208. Determined by heating a 2 gram sample in a 50 milliliter beaker for 24 hours at 302 F (150 C). The heating is carried out in an air-circulating oven.

CTM 0420
 CTM 0002
 0461

CTM 0773. O.K. Bates, "Thermal Conductivity of Liquid Silicones", *Industrial and Engineering Chemistry*, Vol. 41, page 1966, September 1949, units are gm-cal (cm) (sec) (°C). multiply by 241.9 to convert to BTU/(hr) (ft) (°F)

Specification Writers: Please contact Dow Corning Corporation, Midland, Michigan, before writing specifications on this product.

BLENDING

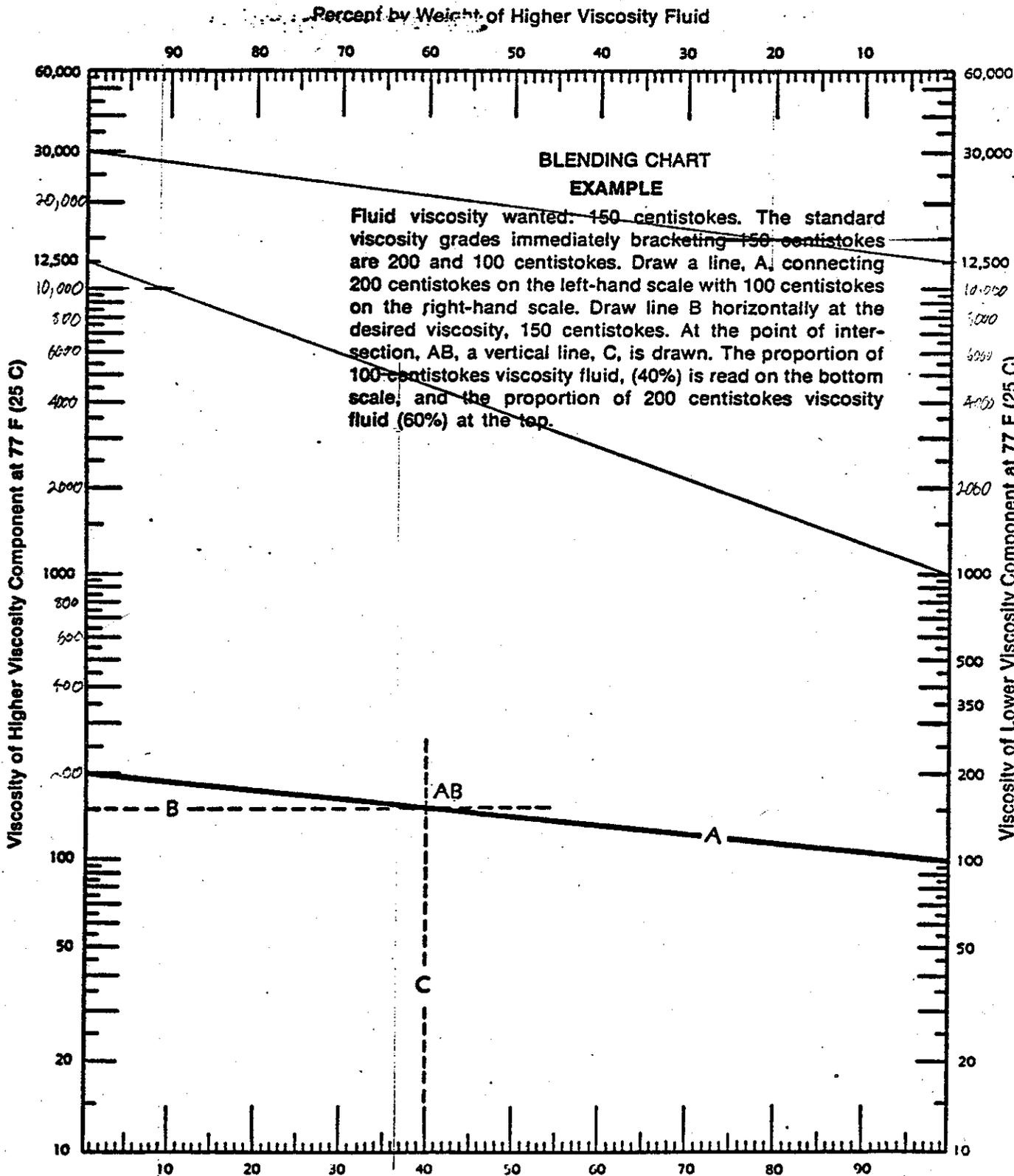
Blending of different viscosity grades of Dow Corning 200 fluid permits any desired viscosity. Although the fluid is available in a number of standard viscosity grades, occasionally an application will call for a fluid of a viscosity between the standard grades.

To use the blending chart: (1) Draw a

line between two points — one on the lefthand scale representing the higher viscosity fluid available; and one on the right, the lower viscosity fluid. (2) Draw another line horizontally across the chart at the desired viscosity rating. (3) Draw a third line, vertically through the intersection of the first two lines. (4) Read off the top and bottom scales the proportions of the available

fluids to blend to obtain the desired viscosity.

Accuracy will be increased by blending the two fluids which immediately bracket the desired viscosity. If a very accurate blend required, it may be necessary to adjust the viscosity of the mixture a second blending.



SHIPPING LIMITATIONS

None for high viscosity grades above 2 cs. DOT Classification: Flammable, for 0.65 and 1 cs. And effective January 1, 1975, 1.5 and 2.0 cs will have DOT classifications of Flammable.

STORAGE AND SHELF LIFE

Dow Corning 200 fluid has an unlimited useful life when stored at 77 F (25 C). Keep 2 cs & below viscosity grades away from heat and open flame.

PACKAGING

Standard viscosities of Dow Corning 200 fluid are supplied in 8-, 40-, and 440-pound containers. All weights, net.

Information and data contained herein are based on information we believe reliable. You should thoroughly test application and independently conclude satisfactory performance before commercialization. Suggestions of use should not be taken as inducements to infringe any patent.

W CORNING CORPORATION
LAND, MICHIGAN 48640

Corning is a registered trademark of Dow Corning Corporation

